Authors' response esurf-2020-20

Inertial drag and lift forces for coarse grains on rough alluvial beds Authors: Georgios Maniatis, Trevor Hoey, Rebecca Hodge, Dieter Rickenmann, and Alexandre Badoux

This document is based on the responses to the comments of the interactive discussion. In addition, there are further clarifications and both the notation and the numbering of the lines here follow the revised manuscript. There were three points highlighted by the reviewers which we focused our revisions on.

- In the previous manuscript there was a general agreement that, although the physics of the IMU measurements were introduced, there was not enough clarity about what an IMU sensor measures and how it is different from other force sensors. This was also the basis for a lot of reasonable questions about how we use the IMU measurements to describe sediment motion. To address this, we restructured all the methodology sections, simplified the notation and introduced a lot of non-technical descriptions in the text (moving all the transformation mathematics to Appendices). In addition, we clarified and simplified the presentation of the drag and lift forces on the tested particles as we describe below.
- The comments from Reviewer 1 lead us to introduce our experimental setup fully. In the previous manuscript we tried to hit a balance introducing the sensors and describing how they can be used (guiding the reader to the thesis of Maniatis 2016 for details). The revised manuscript is a self contained document with all the details about the experimental settings discussed and clarified.
- Reviewers 2 and 3 suggested the introduction of the rotational component of sediment motion in the description. We extended the methodology to introduce the necessary derivations for rotation and we added a new appendix which shows the scale difference between the rotational and the linear (translational) components of transport (Appendix C). We also discuss the differences between the rotation of a sphere and ellipsoid (Appendix C) which makes the description of the Newton Euler model presented in Section 2 more comprehensive.

Additional Revisions, Clarifications and Simplifications

In addition to the comments from the reviewers, we realised that Figure 2 (Figure 3 in the revised manuscript) was misleading. To explain the issue we need to introduce briefly the force balance we use in this work. We describe the net force that mobilises the particle as;

$$\vec{F_{net}} = \vec{f} + \vec{F_{Gr}}$$
(1)

 $\vec{F_{net}}$ is the net force (ΣF in the previous manuscript). \vec{f} is the interaction between hydraulic forces (turbulence) and particle forces (support forces and friction) that is not measured directly by an IMU and cannot be decoupled using IMU measurements. $\vec{F_{Gr}}$ defines the force of gravity.

In the previous manuscript \vec{f} was denoted as F_r and the components of F_{Gr} were denoted as F_{cr} depending on how they are calculated (along the drag or lift direction). There was no clear distinction between F_{net} and F_r which could be very missleading.

For the impulse calculations we account for the forces above the threshold of motion thus for when the norm $F_{net} > 0$. However, the previous version of Figure 3 was suggesting that we consider the impulse of the net force above gravity $F_{net} > F_{Gr}$ which is not correct and has no physical meaning. What we did consider are the impulses of interaction f above gravity both in a scalar form (drag plane/direction) and in a vector form (lift direction). This has a physical meaning because at the threshold of motion ($F_{net} = 0$) the interaction f balances gravity F_{Gr} .

It is possible to understand that we calculate the $f > F_{Gr}$ impulses throughout the previous manuscript because in the corresponding section (Lines 195-207 in the previous manuscript) we show a clear distinction between the components of the f interaction (F_r hydraulic force in the previous manuscript) and the net acceleration components (a_r). However, Figure 2 could really confuse the reader.

To clarify and simplify the presentation we only consider the downstream direction for the drag force in the revised manuscript (while in the previous manuscript we were considering the added forces on the x-y plane parallel to the bed). This has two significant (and we consider positive) effects on the clarity of the manuscript. Firstly, when the reader interprets the force impulse calculations it is easy to relate the drag and the lift forces because now they are both vectors (instead of the version of the previous manuscript where we calculated a vector for the lift direction and a norm for the drag x-y plane). Secondly, we avoid any bias on the statistics and the interpretation of the calculated impulses. Because the particle moves freely the recorded forces have both positive and negative components along each x and y directions. This can complicate the interpretation of impulses as they do not all represent a drag (downstream) motion. For this reason, in the revised version, we only consider the drag component (along the x downstream direction) and we calculate the impulses over the drag positive direction (similarly to how we calculate the impulses over the positive lift direction). This resulted into the following changes on the calculations:

- All the statistics and the graphs of the Results (section 4) that refer to drag impulses are now for the 1D drag force and for the positive direction.
- The fitting of the empirical distribution in the Discussion (section 5.2 in the revised manuscript) for the drag normalised impulses (Figure 7) is different since the drag impulses as defined in the new manuscript are approximated (marginally) better by a gamma distribution.
- In this context, all the fitting and correlation statistics in appendices E and F are now calculated for the 1D drag force as defined in the revised manuscript.
- Finally, the statistics of the 1D- drag force do not support the interpretation of a clear difference between the impulses recorded in the lab and the field for ellipsoid. However, there is an observable increase of the magnitude of the point forces (from the lab to the field). As a result we adapted all the interpretation and discussion points to reflect the drag components as they are defined in the revised manuscript.

It is important to note that the lift calculations are exactly the same with the previous version (where already calculated in a vector one directional form). In addition, this does not affect the introduction of the IMU model and measurements (Appendix A in the revised manuscript). Here, we provide the previous and the revised version of Figure 3, highlighting the main difference and the reason we decided to do this revision.



Previous (misleading) version of Figure 3 in the revised manuscript (Figure 2 in the previous one) The exceedance of the drag force over threshold is depicted as the net force exceeding the threshold of gravity (orange window). This thresholding has no physical meaning and it is a presentation error. By focusing the graphs b,c and d above the green line for the drag force we confuse the reader. The generated impulse we are interested into, and we calculated both in the previous and the revised manuscript, is the impulse of the net force exceeding 0 (green window). This is the equivalent of the f interaction exceeding gravity (see new version of Figure 3, below and Comment 3.7)



New Figure 3 in the revised manuscript (Figure 2 in the previous one) The windows (green and purple) show the exceedance used for calculating the impulses. This can be described either as the net force exceeding 0 ($F_{net} > 0$, green windows) or as the f interaction exceeding gravity ($f > F_G$, purple windows). This is the thresholding used in the previous version as well and this is why the impulse results are exactly the same for the lift force (which is defined as one directional vector in both versions). The drag force is now an 1D vector and not a force magnitude scalar as was introduced in the previous manuscript. Also, in the new manuscript we discuss briefly the noise threshold of the sensor denoted by the black line N.T

Hereafter, for each comment, we have first highlighted the issue, then we provided an answer, and finally we described how the manuscript was adjusted.

REVIEWER #1

Comment # 1.1

The manuscript "Inertial drag and lift forces for coarse grains on rough alluvial bed" presents an interesting method for calculating forces acting on grains in motion in mountain rivers. I think the topic and approach is fascinating as has potential to greatly improve our understanding of transport and flow. My suggestions for improvement mainly focus on (1) explaining more of the conceptual physics and assumptions, (2) simplifying some of the writing, (3) explaining a little more how the flow conditions are perhaps not typical of gravel transport in mountain rivers, and (4) changing figure 1 to illustrate the actual experimental setup of the experiments.

We sincerely thank the author for the thorough review and the supportive comments. We set out to address all the suggestions in the revised version. Hopefully this will become clear in our following responses.

Comment # 1.2

I appreciate the physics framework that the paper is written in, but as a non-physicist I think it would help to have a little more explanation of some of the underlying conceptual ideas (translation into geologese?). Here is an example from the very beginning: To me, the title clearly claims that it explores drag and lift forces for particles ON the bed. After puzzling through it and trying to think through the physics, I don't think this is correct. The methods proposed only apply to forces acting on a particle when it is in active transport, not when it is on the bed. Maybe to a physicist, having "inertial" in the title would make it obvious that the grain isn't actually resting on the bed, but that wasn't the case for me. I am still not sure what the "inertial" part is supposed to tell me. I don't think this is just a matter of semantics; it's a matter of communicating clearly to the geoscience community. Many papers have been written (and that the authors appropriately cite) using force sensors that measure drag and lift forces for grains on the bed. This paper is doing something different, which is unique and fascinating-to measure forces on grains while they are in motion–but those differences should be made clear.

We appreciate the value of explaining the concepts using language that will be familiar to the reader. Some of the underlying concepts we present will be more familiar to people with backgrounds in hydraulics or physics, but we consciously target the audience of this journal (broadly the audience working on geomorphological and geological problems).

The term "inertial" in the context we present primarily means that the reference frame we use to analyse particle forces is static, i.e. that the frame of reference doesn't move with the particle. This is not a common approach in sediment transport but is necessary to enable us to interpret the accelerometer measurements: these are meaningless if untransformed. In addition, the term "inertial" means that, in the reference frame we use, there are no so-called fictitious forces. This observation is very straightforward for the hydraulic forces (and the forces from the surrounding particles) acting on a target particle but needs to be specifically stated for gravity because gravity is a fictitious force for a free moving accelerometer (Appendix A).

Finally, it is very important to clarify that inertial sensors measure forces only when the particle moves. The advantage of this is that the motion of the particle is captured explicitly (the sensor solves a very complicated force balance for us). The disadvantage of this is that there is no way to "decouple" the acting forces without complementary measurements (no information about the hydraulic drag component, for example).

To address this comment (which also overlaps with comments from the other two reviewers) we:

- Restructured completely the methodology sections of the manuscript. Instead of introducing the necessary transformations for the derivation of the accelerometer measurements in the text, we discuss conceptually the measurements and their relationship to the forces applied on the particle (section 2). However, we still provide all the technical background for the necessary transformations in appendices (Appendices A, B and C)
- Simplified the notation and we avoid terms that can lead to confusion. A good example of this was the term "critical force" which we used in the previous manuscript to define the gravity threshold but readers with a background in hydraulics relate to a combination of forces (friction, reaction etc.). This is now discussed and clarified (throughout section 2).
- Explained the difference between accelerometer measurements and other force sensors (such as load cells, Lines 110 -120)
- Introduced a new section in the discussion on the application of IMU sensors in geomorphology (section 5.1)

Comment # 1.3

I suggest adding a paragraph or whatever describing forces on particles from the flow when the particle is stable on the bed, when it is moving but still has its mass partially supported by contact with the bed, and when it is fully entrained by the flow, and what the accelerometer-based calculations actually measure.

In section 2 we clarify further which are the force components captured by the sensor and which are the forces that cannot be decoupled from an IMU. We also describe the rotational component. In the future we hope to perform similar experiments in conjunction with detailed flow measurements and enhance the description of incipient motion.

The threshold of motion (as it is captured by an accelerometer) is discussed specifically in the newly revised section 2.1

Comment # 1.4

Is the way to describe it that the measurements are not forces acting on the particle, but the net force that the particle responds to, from the combination of fluid and solid contacts along the particle boundary?

This is exactly the case; the only difference is that we were using the term "resultant" in the previous manuscript. In the revised version we used the terms net force and net torque to simplify the terminology.

Comment # 1.5

I think that simplifying the language in many places could also make the manuscript easier to read and understand. As part of this I suggest the authors take the time to go through it again and get rid of some superfluous words (such as "proven" in the second line of the abstract, maybe "regime" in the first line, and other places throughout. "has historically been proven" could also be replaced by "is").

We tried our best to address this comment. We have reduced the text and simplified the language throughout the manuscript.

The attached annotated manuscript shows clearly the extend of language revisions. Some distinct examples are:

- Abstract
- Lines 30-45
- Paragraph starting with line 70

- Section 2 throughout
- Appendix A (lines 737 to 745, in-text in the previous manuscript.)

Comment # 1.6

I think the manuscript would be more clear if the authors define drag and lift forces. It helped my understanding to look up physics definitions: drag is simply force parallel to the direction of fluid motion, and lift is force perpendicular to the direction of fluid motion. I did not realize that drag and lift were defined so simply (thanks, I learned something new). I suppose these are sort of defined on line 192, but this is not nearly obvious enough; I did not realize that these were definitions. In particular for lift, I thought it implied differential pressure on the top or the bottom of particles from different velocities (i.e. Bernoulli), and I was confused because it seemed like the authors could not know this since they have no measurements of fluid velocities or pressures. I realize I was wrong, but my point is that drag and lift are terms that bring preconceived ideas about the flow.

The fact that the Reviewer had to go through that enquiry makes the clearer definition of forces necessary. For the revised manuscript we only consider the force along the drag direction r_x (F_{Dnet} in the revised manuscript) and the force along the lift direction r_z normal to the bed (F_{Lnet}). Quoting from the revised manuscript (section 2, Lines 160-166)

"we can use the linear accelerations [*the transformed accelerometer measurements*] of equation (1) to separate the components of the net force (F_{net}) along the drag and lift directions as:

and

"

 $F_{Dnet} = m \ \vec{a_{rx}}$

$\vec{F_{Lnet}} = m \vec{a_{rz}}$

Comment # 1.7

I couldn't quite figure out from the equations whether the water surface slope is accounted for; i.e. are the lift forces the authors calculate actually perpendicular to the mean flow direction, or are they instead parallel to the vector of gravitational acceleration? Figure 1 indicates that they are perpendicular/parallel to mean flow, not horizontal or vertical. Is a small angle approximation required, or embedded in the equations? The resultant lift force (F_{Lnet}) is perpendicular to the mean flow direction (r_z). The confusion was based on the fact that in the previous manuscript we didn't make clear enough that the spherical and the ellipsoid particle had different sensor alignments (but the lift force was always transformed in the static r frame). We now make this clear with a revised figure for the experimental setup (the figure is scaled to fit in this document, Figure 2 in the revised manuscript):



Laboratory setting and initial aligment The diagram shows the arrangement of the bed of hemispheres and the test position (4.5 m downstream the entrance of the flume). Upstream of the hemispheres bed, the flume was filled with densily backed gravel ($D_{50} = 0.015$ m) to allow for the development of the flow. r stands for river-bed (flume in this case) reference frame, b for body frame and i for the inertial reference frame ($_{i}^{r}q = [0.92, 0.14, -20, 028]$). S is the slope of the channel (0.02). The sketch also depicts the initial alignent for each device which (b – i for the sphere and b – r for the ellipsoid).

Comment # 1.8

Another point that I think needs to be discussed, and limitations explicitly pointed out, is that the experimental conditions of the flume experiments are not representative of typical gravel transport Table B1 says that the flow depth was 0.1 m, which is essentially the same as the sphere diameter of 0.09 m. Grains the same size as flow depths are relevant to boulder transport, for example. I can accept the conditions as being informative even if not typical for gravel-bed systems, but it does mean that many of the scientific results, like the relative importance of lift and drag forces, may not be more broadly applicable. The Shields stress is very low for typical thresholds of motion (0.013), which my guess is related to the high protrusion and fact that the grain blocks basically all of the flow depth. Another factor in the flume experiments is probably that the hemispheres mounted on the flume bed only spanned a length of 0.5 m, which means that the boundary layer velocity profile was not anywhere close to developed. Somewhere, the paper should say where within this distance the test particle was placed (i.e., what was the distance between the upstream edge of hemispheres and the starting position of the test particle?).

We answer two comments here because they relate. In the previous manuscript we did not provide enough detail for the experimental regime, guiding the reader to the thesis of Maniatis 2016. For example, we only reported the mean critical discharge for all the experiments (approximately $30 \, l.s^{-1}$) which is not complete because the ellipsoid and the sphere have different thresholds. In the new manuscript we describe in detail both the experimental setting and the measured or calculated metrics. We also state clearly that the experiment is only relevant to large grains and high exposure to the flow (Figure in Comment 1.7). Here are two quotes from the revised manuscript that describe the initial position and the hydraulics for the flume experiments (section 3.1):

"The hemispheres were glued to form a 0.5 m (L) x 0.9 m (W) section and the whole section was placed at the point which allowed for the test particle to be at 4.5 m from the upstream boundary of the flume. A thin layer of 1.5 mm uniform sand was glued to both the hemispheres section and the upstream flume surface. The section upstream of the hemispheres was filled with very densely packed non-uniform, rounded gravel (D₅₀ = 0.015 m) enabling the development of turbulent flow. No sediment transport occurred from the upstream gravel section during the entrainment experiments. The hemispheres were glued in positions that produced a 0.045 m protrusion of the sphere above the top of the hemispheres, although the sand layer made the protrusion higher, by partially filling the gaps between the hemispheres, and non uniform around the test particle (≈ 0.050 m). The ellipsoid was only supported by the hemispheres and was fully exposed to flow (Figure 2."

"The critical discharge for the sphere was $24.8 \pm 1.8 \text{ l.s}^{-1}$ which corresponds to a critical depth of 0.095 ± 0.015 m, measured from the top of the supporting hemispheres. The critical discharge for the ellipsoid was $45.2 \pm 2.21 \text{ s}^{-1}$ which corresponds to a critical depth 0.12 ± 0.02 m ($\tau_* = 0.01$ and 0.02 respectively, $\tau^* = \frac{\text{H S}}{(\rho_p - \rho_f) \text{ D}}$, Appendix D) (...) The flume experiments have particle diameter to flow depth ratio close to 1 (d/H \approx 1) despite the fact that the particles were fully submerged at the critical depth for all the experiments (Figure 2). The tested conditions are relevant to the entrainment of coarse particles in steep mountain streams but they should not be

directly generalised to other bedload transport regimes despite the generality of the Newton-Euler model presented in section 2."

Comment # 1.9

Similarly, it is unclear whether the result of lift and drag becoming uncorrelated from the lower flow (flume) to the higher flow (field) conditions represents the difference in flow depth relative to grain size or the difference in flow intensity. Also, would be helpful for Table B2 to have a calculation of tau* for comparison.

For the first part of the question, we observe a generally weak correlation between the net drag and lift forces. The highest correlation was observed for the spherical particle in the flume (R=0.3). For the ellipsoid the correlation is consistently weak (from R=0.046 in the lab to R=-0.0055 in the Erlenbach during transport). The correlation statistics are given in Appendix E. To answer why the correlation becomes even weaker in the field we would need a series of incipient motion experiments in the field and that was not possible. For the conditions we captured, the sensor was constantly in tractive motion after released. However, in the discussion we refer to the work of Shih and Diplas, 2018 (and other works) and state that the differences in turbulence will have a very strong effect on the applied forces. For the second part of the question, we now provide τ^* and have removed the metrics of Table 1 that rely on τ_b which complicated the comparisons (highlighted also by Reviewer 2). Tables B1 and B2 were also combined (Appendix D).

Lines 420-425 of the revised manuscript: "...Differences in force magnitude and duration can relate to transitions, as described by Shih and Diplas (2018), from hydraulic "impulse controlled" transport, as in our flume experiments, to "force-magnitude controlled" transport corresponding to the dynamics recorded in Erlenbach. However, an important difference between laboratory and field experiments lies in the scales of turbulence (Coleman and Nikora, 2008; Singh et al., 2009), which requires further investigation since detailed flow measurements were not made during the presented experiments (eg. PIV measurements)."

Comment # 1.10

I am a little perplexed by the measured particle protrusion of 0.045 m, i.e. half of the grain diameter, of both the experimental sphere and also the hemispheres. This means that the test particle was basically resting on the flume bed (or very close to it) in addition to resting on four surrounding hemispheres, right? And it also means that the hemispheres were not spaced as closely as possible, but spread out in order to allow the test grain to be down low, right? Figure 1 is a conceptual diagram that does

not match the actual experimental design. This should be changed so that Figure 1 is drawn to match the experimental conditions. The experimental grain should be on a bed of hemispheres, not full spheres, the spheres should cover a length of 0.5 m (so I guess 5 hemispheres?), and the spacing and placement of the test grains should be appropriate so that it reflects the actual grain protrusion. Table b1 says the protrusion is half of the particle diameter, which means the particle was essentially resting on the flume bed? I also suggest indicating the approximate water surface on the figure, since it is basically at the top of the test particle.

Table b1: was the protrusion the same for the ellipsoid?

We reply to both comments in the responses for Comment 1.7 and Comment 1.8 and the revised section 3 of the paper.

Comment # 1.11

Line 85: I'm not sure its useful for the authors to give their opinion that not being able to measure position has significantly limited the IMU use. That may be true but there isn't any way to know if that is the main reason, and does it matter? The current paper doesn't solve this problem, it presents a new way to use the devices for another problem. Also, I think the tone of this paragraph is unnecessarily dismissive of previous work done using similar devices. I don't know what "best considered to be preliminary" is trying to say, other than to belittle this work. The authors come across as arrogant. To me, these works show that there is a benefit and potential of pulling different kinds of information from instrumented particles.

As we stated in the online discussion, we had no intention to dismiss previous work. We just believe that there is still a lot to learn about those sensors and we (as a community) need to be more aware about the details (and the physics) of how they operate. We revised the sentence and we added the following paragraph in the newly added 5.1 section of the Discussion

"Further, there has been a recent rapid increase in use of IMU sensors, but most offthe-shelf IMU sensors are not suitable for the range of forces characterising natural sediment transport, especially if the focus is on particle interaction or impacts (Maniatis et al., 2013). In addition, the physics of IMU sensors are complex and a number of common assumptions about their use do not always hold. For example, while deadreckoning appears to allow positions to be recovered by double-integration of linear accelerations, uncertainties introduced during the production of IMUs (mostly nm scale imperfections on the alignment of the sensors) lead to extreme uncertainty in positional estimates. A second issue involves calibrating IMU accelerometers which has often been done using free fall drop experiments. An accelerometer in free fall will measure 0 acceleration despite being subjected to the acceleration of gravity, as gravity in the context of the body frame of the accelerometer is a so-called fictitious force (Appendices A and B). Consequently, the force/impact results of a free fall drop experiment which relies solely on IMU measurements are highly dependent on how quickly the sensor is programmed to enter and wake up from the free fall detection state (Clifford, 2006). It is possible to approximate the height of the free fall using the approximate time of the free fall state. However, the measurement of the impact force needs a very detailed description of both the impact surface and the low-level code that controls all the basic operations of the sensors (on-off routines, logging, storage-handling etc.). This low-level programming is a black box for proprietary off-the-shelf sensors and for users who are not trained programmers. Finally, it is not possible to derive directional information, even for forces, for long mobile periods without complementary corrections (Kok et al., 2017) or without a detailed presentation of the reference frames involved and their initial alignment."

Comment # 1.12

Lines 199, 202: I think the critical drag and lift force equations should have their own equation numbers, even though they are simple equations. They are important for understanding the analyses, and it was annoying to have to go back and hunt for inline equations.

We have revised the notation (and the terminology) to avoid confusion and the same parameters are now defined as the gravity components. Exactly the same definition with the previous version but different name so it is clear we only refer to the gravity force. They also have their own equations. From the revised manuscript (section 2):

"The vector $\vec{F_{Gr}}$ defines the force of gravity rotated in the r frame and compensated for hydrodynamic effects (...). The magnitude of $\vec{F_{Gr}}$ components along the drag direction (r_x) is given by the vector:

$$F\vec{G}_D = F\vec{G}_{Tx}$$

For the direction normal to the bed (z_r , direction of lift hydraulic force) the $\vec{F_{Gr}}$ component is the vector:

$$\vec{FG_L} = \vec{F_{Gzr}}$$

Comment # 1.13

"

226: Why 50 hz? More to the point, is there a physical argument that this sampling rate is sufficiently fast to capture the forces acting on the grain? In any case, the authors need to explain why this sampling rate was chosen. Would a slower or faster rate also work?

We have now added the reference that lead us to chose this sampling rate (section 3).

"The nominal sampling frequency of the sensor (used for all the flume and field experiments presented in this work) is 50 Hz which permits constant use for approximately 5 hours (LiPo rechargeable battery). This frequency was chosen after considering displacements of 15 particle diameters per second reported by Drake et al. (1988). This is adequate for capturing the dynamics of the particles if collisions and strong interactions with the bed are excluded (different type and higher frequency piezoelectric sensors are more suitable for the full measurement of impacts)."

Comment # 1.14

215: I think this point should be made more prominently, and explained, earlier in the paper: that the fluid is not being measured. When I got to this point I became confused; I was still assuming that the paper would compare fluid measurements and forces on the grains, and it did not make sense that the fluid part was ignored.

We start section 2 of the revised manuscript with an elaborate (but not technical) description of what an accelerometer measures to define the type of recorded forces. Also, we introduce separate notation for the forces that are captured directly and estimated indirectly (capitals and low case letters respectively) to highlight the differences even further. Example sentences from section 2 that highlight this in the revised manuscript are:

"If the sensor moves and the sensor frame accelerations are compensated for gravity (by removing gravity from the raw accelerometer measurement) then the sensor will record the 3D components of the resultant or net force that mobilises the particle. This resultant is the force that can be observed from an observer who is static in relation to the particle."

"...The left hand elements of equations 1 and 2, describe the forces applied on the particle. Hereafter, the lower case vectors and scalars (e.g terms \vec{f} and $\vec{f_{rot}}$) will refer to the interactions between hydraulic forces (turbulence) and particle forces (support forces and friction) that are not measured directly by an IMU and which cannot be decoupled using IMU measurements."

Comment # 1.15

233: 0.5 m long is very short relative to lengths needed to develop boundary layers (i.e. velocity profiles). How much distance was there between the test particle and the upstream end of the hemispheres? I presume that the upstream flume surface was planar other than the 1.5 mm sand? It seems to me that the possible effects of this this should be addressed or at least acknowledged somewhere.

The hemisphere bed was 4.5m downstream from the flume entrance (see Comment 1.7 and Comment 1.8).

Comment # 1.16

245: Give a little more explanation than just "section 4" for where these numbers came from. (in the caption to fig2 it says it used equations 9 and 10).

This is now the corresponding section 2.1 which is much smaller and easier to follow. We refer to the gravity components we describe in Comment 1.12.

Comment # 1.17

252: Give citations and a more complete explanation of why entrainment is defined by this displacement.

We do not think there is a need for a citation because the incipient motion threshold is explicit for this work (is for $F_{net} > 0$). The clarification about how we define the entrainment as the full dislodgment of the particle is given in section 3.1 of the revised manuscript.

"We define the entrainment point as the initiation of the vibration that dislodges the particle by one particle diameter."

COMMENT # 1.18

265 or around: I realize the slope is in table b2 but somewhere here say the key numbers, especially slope of 0.1 and flow depth of 0.15 m. Also describe something about the pocket or pockets the grain was placed. In the text says on a step, but explain what that means. Resting on 3 or 4 grains of similar size?

In the revised manuscript (section 3.3) we include key numbers and we clarify that the sensor was released over bare bedrock and that there was no initial position relevant this experiment. "...The stream has a step-pool morphology allowing the sensor to be retrieved from pools, so the ellipsoid sensor was submerged on a bare bedrock section close to the edge of a step (average slope S = 0.1, cross-averaged flow depth H = 0.1 m, τ_* = 0.095), aligned to the same orientation as the riverbed (...) and allowed to be transported until it stopped moving and remained immobile for at least 10 seconds (...). In all the experiments the sensor was entrained fully and immediately as there was no vibration *in situ*. The first one second of each transport event was removed from the data as the effect of holding and releasing the sensor were still present."

Comment # 1.19

Figure 2: in panel e, I think the Flcr and Fdcr lines are flipped. In panel a, the part labeled "vibration" really represents grain transport and rotation, right? See also line 291. To me vibration implies a grain wobbling or rocking back and forth in place. How would you define grain vibration as similar or different from grain transport? Is grain vibration just being defined as just transport over a distance less than one diameter? A grain rotating as it is transported?

Figure 2 is now Figure 3 and has been revised extensively to clarify the differences between that applied and the resultant (net) forces. FLcr and FDcr have been replaced (in terms of notation) with the FG_D and $F\vec{G}_L$ gravitational components (Comment 1.12). In addition to the explanation we provide in the online discussion (the magnitude of the gravitational components for a given direction is not always intuitive), we only accounted for magnitudes in the previous manuscript along the x-y plane. This is now should be clear after the revisions we describe in the beginning of this document. The label "vibration" refers to pre- full dislodgment vibrations which are also captured by the sensor (from our definition of entrainment, Comment 1.17).

Comment # 1.20

It seems a little surprising to me that at the end of the paper, the analysis and argument is made that lift forces are more important to entrainment than drag forces, but in Figure 2 panel A and D the drag forces clearly exceed their respective thresholds much more than do the lift forces. I don't know if it is just these examples that are shown or that I'm interpreting something wrong.

Figure 3 (former 2) has examples from both the sphere and the ellipsoid in both flume and field conditions. Also in the previous manuscript the drag force was the absolute value of the addition of 2 forces. In the revised manuscript we have simplified the presentation by calculating the 1D component. When we collate the force data from the ellipsoid (Forces in Appendix E) it becomes clear that the effect of the

lift component is more pronounced for the ellipsoid than for the sphere (for both the laboratory and the field experiments). In addition, in the revised manuscripts we clarify what the "respective thresholds" are. In the previous version of Figure 3 there was an ambiguity which could miss-lead the reader and is now clarified (see introduction of this document)

Comment # 1.21

I'm not sure what "scale difference" means. It seems to me that the main difference is that in the flume experiment the particle was resting stably prior to entrainment, and this entrainment was analysed. In the field case, the particle was just always and fully transported, and so the data just represent particle transport over a rough bed without any data on the particles going from resting to mobile. Also the hydraulic conditions were different.

This is a point also raised by Reviewer 3. Our intention was to point out that in the field we should have different turbulence scaling than in the lab, and that this could partly explain the behaviour of the ellipsoid and specifically the dependence on the lift component which is even more pronounced in the field. We briefly discuss the differences in turbulence in the discussion section (Comment 1.9)

Comment # 1.22

Explain how you know that the distributions are heavy tailed. I presume this is coming from the Weibull, gamma, lognormal fits presented later in figures D1 and D2? If you're going to say heavy tailed in the results section you have to explain it there. You could just remove this mention at this location I think without losing any understanding.

This is purely observational from the histograms of Figure 6 (former 5). The fitting in the appendices concerns the calculations presented in Figure 7. We re-phrased to "right -skewed".

Comment # 1.23

Figure 3b, right panel: the blue line (lift) looks like an odd or poor fit to the data, because there is a whole cluster of blue data points well above the line. Are there a lot of blue points hidden by the red points? Plot the data in some way that data points are not hidden, such as smaller symbols or open symbols.

The points are much smaller in the revised version and they overlap less. Thus it is obvious that there are points close to 0 that justify the fit. This is also confirmed by the histograms for each comparison.

Comment # 1.24

I realize that this is not the subject of this paper, and I am not sure the authors have enough data to really figure this out, but it would be interesting to know how well cumulative impulse from a given hop scales with transport distance of that hop.

This is very interesting but to calculate hop distances using our measurements we would need to derive distance from accelerometer measurement and that is very unreliable at the moment. We can workaround this with video recordings or other types of tracking. We plan experiments where position is calculated externally which could address this point.

Comment # 1.25

A final point that I think the authors should acknowledge is that these calculations are untested. They do not know how correct these force measurements actually are. I think that is fine as long as it is stated; I would suggest saying that future work should explore and try to validate the accuracy of these measurements in some way.

We address this in text (Discussion 5.1)

"Here, we calibrated and deployed a commercial IMU sensor following standard procedures (Maniatis, 2016), but the precise corrections used are sensor specific and similar procedures should be followed again for any other IMU sensor. The calibration of force measurements is likely to be standardised and simplified in the near future as the use of IMU sensors develops further. Similar standardisation for the direction of forces is potentially further away as it requires using IMU sensors that rely on optical technology and which are currently not manufactured with physical dimensions or within a price range that is accessible for sediment transport studies (De Agostino et al., 2010)."

COMMENT # 1.26 PRESENTATION / GRAMMAR AND WORDING: REVIEWER 1

- Line 25: suggest simplifying wording of "multi-variate two-phase flow defined by a range of interacting complex subprocesses. . ."
- Line 34: change "it's" to its
- 335: reword "the on the"
- "Extended analysis" is not a very informative section title; suggest changing to something different.
- 368: "of of"

• Stylistically, I suggest combining lines 114-128 into one paragraph; I think it is better than having four very short paragraphs.

All the presentation and grammar comments have been addressed in the revised version (see attached track changes manuscript)

REVIEWER # 2

Comment # 2.1

The manuscript by Maniatis is based on a study that uses two custom-built smart particles (one spherical and one elliptical in shape) that contain an accelerometer and gyroscope. The manuscript focuses on (1) presenting a method for putting the accelerations extracted in the particle's frame of reference to one from the perspective of a fixed observer and (2) presenting the lift and drag forces measured while the particle was in motion in a laboratory and field experiment. The paper's strengths are the presentation of the mapping method between the two frames of reference and the introduction of the idea of inertial drag and lift impulses. The inertial drag and lift impulse are similar to the impulse idea put forward by Diplas et al. 2008 (Impulse = integral of force with respect to time applied to a particle) but is based on particle forces inferred from particle motion rather than particle forces inferred from fluid measurements. I believe the paper presents (1) useful methodology information for those using accelerometer data in particles, and (2) useful data pertaining to particle motion. At the same time, I have a few questions for the authors and some suggestions on the presentation of the work that I think should be addressed.

We sincerely thank the Reviewer for the thorough review and the positive comments.

COMMENT # 2.2

My primary suggestion, or concern, is that the authors make it explicitly clear that all of their force measurements (and impulse calculations) require that the particle be moving (if I am understanding things correctly). It would also be helpful if they expanded their discussion on the benefits and limitations of such measurements. Many of my presentation suggestions below reflect this desire for it to be clear that the forces measured are only those extracted from the particle accelerations.

Those are points we now highlight throughout the manuscript. Firstly, we define the forces in a clearer manner and we indicate the interactions that are not captured by the accelerometer (interactions f). Secondly, we clarify the threshold of motion (see introductory statement and Comment 1.20. Finally, we discuss the pros and cons of the method in the discussion (new section 5.1). From the revised manuscript:

Comment # 2.3

"The advantage of using an IMU sensor for capturing grain motion is that the sensor apparently solves a complex force and torque balance and removes any ambiguity for whether or not a test particle is in motion, as motion leads to the explicit thresholds F_{net} and/or T_{net} exceeding 0. Entrainment is captured directly and, assuming correct sensor calibration, robustly. IMUs can be a useful tool for geomorphologists since they offer a realistic prospect for monitoring particle motion during transport without invasive apparatus which is not possible with standardly used equipment, especially in field applications (e.g. PIT tracers). At the same time, it is important to recognise that that exceedance the explicit thresholds above does not always produce complete dislodgment of the particle and also does not directly describe the modes of transport in the context that is commonly assumed for sediment hydraulics (...). For a complete understanding and effective prediction of grain motion both the hydraulic and the particle forces need to be measured, analysed and decoupled from the inertial forces we measure in this study. "

COMMENT # 2.4

It seems that you have the highest number of entrainment events for an inertial impulse of zero. Is this because the primary motivating impulse came at a time before that which could be measured by the particle?

This is difficult to answer with certainty. The zero impulses are generated from forces that reached the threshold value and did not exceed it for more than 1/50th of a second (the sensor's sampling frequency). As we clarify in the revised manuscript, the threshold value for the drag direction (now the x direction and not the x-y plane) is the the noise threshold of the sensor as we only consider the norm of the net forces exceeding 0. For the lift direction is the component of gravity that balances the f_L interaction. Also, not every exceedance dislodges the particle and we define the entrainment point at the start of continuous 1 particle of motion. Particle dislodgement is still stochastic for the inertial measurements. There is a very high chance that the motivating hydraulic impulse that begins the mobilisation of the particle comes before the particle starts to vibrate but we have no way to verify that with the existing dataset.

COMMENT # 2.5

Along these lines, do the potential travel paths of the particle dictate the forces measured? That is, are the FL and FD measurements sort of pre-determined by the orientation of the particle relative to others in the bed?

 F_{Lnet} and F_{Dnet} (consequently and the interactions f_L and f_D) would be different in different bed topography, and so would be altered in a different microtopography.

We have highlighted in the revised manuscript that the results are highly dependent on the experimental settings (Comment 1.8). At the same time, it is difficult to distinguish the most dominant control on the resistance of the particle (the orientation of the particle, the surrounding particle forces or the inertia of the particle). The orientation of the particle and the pivoting angle will dictate the mode of entrainment because the boundary condition will be different. At the same time, it is fair to assume that the trajectory of large particles will depend a lot on their moment of inertia (I_{cm} in the paper) which describes the distribution of mass within the body frame of the particle and affects the direction of the resultant force. This is an exceptionally interesting question that will require a different data set in order to be answered.

COMMENT # 2.6

In equation 5, where is the contact force with the bed? Is it tied into the gravitational force terms? The critical drag and lift forces will depend on the submerged weight of the particle and the orientation of the grain within a pocket through the contact reaction force. The orientation of this force will also influence where it is more likely to have lift or drag dominate. How does it factor into equations 9 and 10? It should, shouldn't it? And 5. Along with the preceding question, how is this contact force for accounted for throughout the range of a particle's motion as it moves and interacts with the bed?

Along with the preceding question, how is this contact force for accounted for throughout the range of a particle's motion as it moves and interacts with the bed?

In addition to our answer in the interactive discussion, we now clarify that the interaction f (denoted as F_r in the previous manuscript) is the interaction between the hydraulic forces and the resistance or reaction forces. We realise that the previous manuscript gave the impression that we isolate somehow the hydraulic force but this is not possible to be done with and IMU (Comment 1.14 and revised section 2 in the manuscript).

Comment # 2.7

Please provide a figure showing the experimental setup with flow depth and bed arrangement. I would think that τ_b has little meaning in terms of mobilization with your particular laboratory setup.

We provide a detailed description of the experimental setup in the revised manuscript (Comment 1.7). We also removed all the τ_b calculations from the manuscript.

Comment # 2.8

Terminology in Figure 2 and elsewhere. When do vibrations turn into motion? Does entrainment start when the particle reaches a distanced travelled of 1 diameter, or does entrainment start when the particle starts to move and then continues on a path that leads to it moving 1 particle diameter? Also, what is a non-entrainment event with a measured inertial impulse? Does that correspond to a case where the particle started to move out of the pocket but then fell back down before reaching the apex?

We explain in the begining of this document that Figure 2 included a mistake which could confuse the reader. We define entrainment at the point of full dislodgment (Comment 1.17). As a result, during a vibration period the particle moves in its pocket but is not fully dislodged, as suggested by the Reviewer.

COMMENT # 2.9

In the discussion, I'd suggest non-dimensionalizing the force (maybe using sub- merged specific weight) when making the comparisons to other work. Also, how do your inertial forces compare to standard drag estimates using velocity, particle size, and a drag coefficient? What types of relative fluid/particle velocities are needed?

We now provide a non-dimensionalised force values in the discussion (Section 5.1, Lines 395-410). The comparison of inertial forces with standard drag estimates is more difficult because most of the literature is devoted to fully developed flow which is not the case for our experimental setting (small bed of hemispheres and grain diameter/ depth ratios close to 1).

COMMENT # 2.10 PRESENTATION QUESTIONS/ CONCERNS/ SUGGESTION FROM RE-VIEWER 2

The paper is reasonably well written, but I do have some suggestions below that I think would help to improve the presentation of the work.

Abstract:

- L.1 Delete "been"
- L.4 Replace "on sediment" with "on a particle during"
- L.5-7 The sentence "Today, twenty years.... for the issue" is not needed in an abstract. Suggest deleting it
- L.9 Change "grains on" to "grain moving on"

• L.11 Change "resulting to the" to "resulting in a"

All those comments have been accepted and addressed. See attached track changes manuscript.

Comment: Introduction:

- overall I think you can shorten down the introduction
- L.38-41 I think you can remove the sentence that starts with "The term Lagrangian..." Most people know the difference between a Lagrangian and an Eulerian frame of reference

As we said in the online discussion, we tried to keep the balance between being concise and explaining all the terminology to avoid confusion.

- L.44 Delete "(the exact definition of turbulence impulse)"
- L.72 Change opening sentence to: MEMS-IMU sensors ideally measure forces at the center of mass
- L.74 change to "acting on the grain as it moves"
- L.83 suggest changing to, "...and electrical engineering. Modeling of IMU error..."
- L.91 and 92 suggest changing resolve and resolution to "map" and "mapping" or "rotate" and "rotation" Frames of ref

- L.123 Suggest, "... frames is non-trivial. A widely-used method..."
- L.124 Suggest changing "to apply" to "the application of"

All those comments have been addressed as suggested by the Reviewer.

Comment: L.129-end of the section. I'm torn here. I could see all of this being better suited for the appendix if the focus of the paper is on the data from the particles. However, if you want the paper to be about the mapping between the frames of reference then you should keep it here.

We followed the Reviewers advice and moved all of the tranformations to appendices

- L.157-158 Delete the beginning of the sentence that spans the text "As sediment" to "linear acceleration". Just start with ar. Inertial measurements
- L.211 change to Ti
- L.216 suggest changing "that mobilizes the particle" to "acting on the particle once it starts to move" Lab and field experiments
- L.238 0.028 l/s is a discharge, not a rate of increase in discharge
- L.240 change "recorded" to "video of"
- Figure 4. If I'm correct, I think you reference figure 4 before referencing figure 3. It should be the other way around
- L.254 suggesting changing to, "...inertial impulses for cases were the grain started to move." Discussion
- L. 360 I'd suggest making the extended analysis part of the discussion. Use subsection for the different components of the discussion such as comparison with other work and L.368 delete one of the "of"s

All the comments have been addressed according to Reviewer's suggestions.

REVIEWER #3

Comment # 3.1

This manuscript is intended to quantify the dynamics of entrained sediment particles by using advanced inertial force monitoring systems. I believe this work has good contributions to this long-standing problem and it is worth being published. Overall, the materials are properly organized and relevant concepts explained concisely. I see no significant problems for its publication, but there are several minor suggestions for the authors to consider in their revision. Below, I listed my comments and suggestions as I read through the manuscript.

We want to thank the reviewer for the thorough review and the supportive comments. We did address all the suggestions in the revised manuscript as will hopefully become clear in the following responses.

Comment # 3.2

Regarding the Lagrangian approach, Ballio et al. (2018, "Lagrangian and Eulerian description of bed load transport") provide a unified framework to describe sediment particle motion under different frames of reference and quantification methods. The authors may want to refer to this work and highlight their particular contributions to the topic.

As we mention in the online discussion we consider this to be an important aspect of this work. The following paragraph in the discussion (section 5.2) discusses the topic and the work of Ballio et al. 2018

"This work uses a theoretical framework which has the potential to enhance the mathematical modelling of sediment transport. The Newton-Euler model of section 2, in conjunction with the quaternion transformations of Appendix A, can be read as a 3D and unrestricted Lagrangian - Eulerian model for sediment transport. In our analysis, particle dynamics are transformed from a Lagrangian domain (and the mobile body frame of the particle b) to a static Eulerian domain (frame r) which is most commonly used for the analysis of turbulent flow. Ballio et al. (2018) analyse the topic in detail and provide a comprehensive 1-D Lagrangian - Eulerian model which also accounts for the intermittency of sediment transport using a binary classification of mobile and non-mobile states. Our presentation can be used to define 3D Lagrangian dynamics, including rotation, in full and then to transform the corresponding kinematic properties to the Eulerian domain for direct comparison with the turbulent forces. We acknowledge that the verification of the 3D Lagrangian - Eulerian model is heavily dependent on the inertial measurements, and particularly the constant tracking of relative orientation between the frames. However, it is possible to predict that future calibration experiments deploying IMUs will be used this way to parametrise simulations."

COMMENT # 3.3

L58: The issue of over-prediction of transport rates was also extendedly discussed by Bunte et al. (2004, "Measurement of coarse gravel and cobble transport using portable bedload traps") and Singh et al. (2009, "Experimental evidence for statistical scaling and intermittency in sediment transport rates"), and other related papers. The reference can be updated with these contributions.

We accepted the comments and added the references in this sentence (Lines 55-56 in the revised manuscript).

Comment # 3.4

Sec. 2 (Frames of reference, rotations and IMU measurements) provides details of mathematical expressions for the conversion/transformation among different frames of reference. But Fig. 1 needs some more clarity on the specifics of these frames, e.g. (x,y,z) (rx,ry,rz),(ix,iy,iz).

The figure has been revised to explain the experimental regime better (Comment 1.7). In addition, we provide an extra figure in Appendix A, dedicated to the frames of reference in the context of inertial dynamics.

Comment # 3.5

Based on their formulation, Eq. (9) only concerns the threshold condition in the tractive mode towards downstream, and Eq. (10) the upward movement. Yet, when approaching the strictly critical condition, particle rolling, which has even lower resistance, can be the most predominant entrainment mode. The authors can consider adding an extra formula describing the rolling threshold for providing a more complete framework.

We introduce the threshold of motion in the revised section 2. We also explain the differences in the magnitude between the tangential force applied on the surface of the particle (responsible for rotation) and the linear forces applied on its centre of mass. Finally we explain why the rolling of rigid body isn't generally the same with the rolling mode of entrainment (or rollover). From the manuscript: "The torques are analysed in the body frame of the particle (b) and a non-directional description of the rotation threshold is given by the norm of angular accelerations $(\|\vec{\alpha_b}\|)$ exceeding 0. (..) the critical condition for particle rotation is given by:

$$I_{net} = I_{cm} \|\vec{\alpha_b}\| = I_{cm} \|\frac{d\vec{\omega_b}}{dt}\| = f_{rot} \ R \ge 0$$

(..) Using the above equation, it is possible to estimate the magnitude of the tangential component f_{rot} (Appendix C). The calculation reveals that, for the scale of the particles discussed here, the effect of the tangential force, in terms of force magnitude applied on the particle, is negligible in comparison to the linear forces. For this reason, we will only focus on the linear net force and the interaction \vec{f} which we calculate as the difference between the net force and the gravitational components. However, we demonstrate the scale difference between the rotational and the linear components of particle motion in Figure 3 and Appendix C.

(..) the total kinetic energy can by calculated from these velocities as $K = \frac{1}{2}m\|\nu_r\|^2 + \frac{1}{2}I_{cm}\|\omega_b\|^2$ with $\frac{1}{2}m\|\nu_r\|^2$ being the translational and $\frac{1}{2}I_{cm}\|\omega_b\|^2$ the rotational component. The condition K = 0 represents the threshold of rolling for any rigid body. However, this is not a direct equivalent to the typical rolling mode of entrainment because of the differences in the definition of rotation (spinning vs orbital) and the fact that here there is no assumption about the slipping condition as K can be used to describe a rolling with slipping motion"

COMMENT # 3.6

L234. Has the 1.5 mm uniform sand glued to surface or also movable?

No, the bed was washed before the experiments and the sand was static. Also there was no sediment transport from the upstream coarser section (Comment 1.7)

Comment # 3.7

Fig.2 (b) describes the change of drag forces during the noted five entrainment events. The pattern shown here, however, somewhat contradicts the impulse model mentioned later in the manuscript (e.g. L278-279). If all these five events follow exactly the impulse criterion, the events of higher magnitude should persist for a shorter duration, and vice versa, to maintain at the same impulse level. In other words, these events, very likely, represent different degrees of particle mobility and not the most extreme cases of entrainment (minimal critical impulse). I think this aspect is worth mentioning in the discussion.

This comment reflects both the confusion that could cause the previous version of Figure 3 (analysed in the first section of this document) and the fact that the IMU measurement needed to be discussed more. The Reviewer is correct: there are different degrees of mobility represented which are "masked" because in the previous manuscript we added the forces from two directions. In the revised manuscript we perform a vector calculation, which should clarify how the particle behaves and how the minimal impulse is captured (see first section of this document and revised Figure 3).

Comment # 3.8

The calculation of entrainment probability is not clear to me. Is it described as the ratio of entrainment duration to the total observation time, or the ratio of entrainment events to the total exceedance events observed? I assume it is the latter definition. Adding an equation will help to clarify this point.

The latter is the case (entrainment / exceedance events) but instead of using the ratio we perform a logistic regression (as also used in Maniatis et al., 2017). We believe that with the clear definition of entrainment and applied forces, we can avoid adding one more equation to this manuscript.

COMMENT # 3.9

L300. The scaling effects are usually attributed to the intermittency of particle motion dictated by turbulent flow structures. Yet, this sentence seems to suggest the role of physical scale between the laboratory flume and field stream. A clarification will be helpful (see discussion in Singh et al., 2009).

We intended to refer to the differences in turbulence (non-fully developed flow in the lab, developed (but still shallow) flow in the field). We clarify that in the discussion of the manuscript and in Comment 1.9.

Comment # 3.10

Comment: Fig. 4(a) can be improved by using different shading colors for FD and FL, respectively.

We did follow the Reviewer's advice.

The differences in magnitude of critical drag and lift forces between this work and literature data are attributed to the particle sizes, or corresponding mass, used differently. To resolve this issue, a dimensionless quantity, e.g. FD/particle weight, FL/particle weight, can be considered for both the present work and previous reports.

This point was raised by Reviewer 2 and we now provide the dimensionless quantities in the discussion (Section 5.1)

L325-332. The description of the negative lift forces can be more precise in relating to the threshold of motion conditions. Specifically, when the negative lift force appears, the probability of particle lifting reduces, and also, the resistance to the tractive movement increases via the enhanced intersurface friction.

We considered this comment, but since the interaction f in our work is not decoupled (it is the interaction between hydraulics, contact forces and friction) we decided to avoid a reference to friction because it could mislead the reader. We will hopefully address this comment in a future work.

Comment: L353-357. The description of the entrainment mode of rolling can be placed in the earlier section (see the previous comment) to avoid confusion.

We address this comment in Comment 3.5

In the rest of this document we present a fully annotated version of the new manuscript with all the changes highlighted as well as the output from the Acrobat Reader Compare Files tool. We used the tool to capture figure insertions and other stylistic changes that are not always captured with latexdiff (or they cannot be complied easily). The vast majority of the annotations must be visible in any pdf reader, but if the editors encounter problems reading the document please feel free to contact GM.

23/07/2020, 19:25:46

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Styling and Annotations

75 Styling 217 Annotations

Go to First Change (page 1)

Inertial drag and lift forces for coarse grains on rough alluvial beds measured using in-grain accelerometers

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Abstract. Quantifying the force regime that controls the movement of a single grain during fluvial transport has historically proven to be difficult. Inertial Micro Mechanical and Electrical Sensors (MEMS) sensor-assemblies that mainly comprise micro-accelerometers and gyroscopes) can used to address this problem using a "smart-pebble": a mobile Inertial Measurement Unit (IMU) enclosed in a stone-like assembly that can measure directly the forces on a particle during sediment transport.

- 5 Previous research has demonstrated that measurements using MEMS sensors can be used to calculate the dynamics of single grains over short time periods, despite limitations in the accuracy of the MEMS sensors that have been used to date. This paper introduces and tests a theoretical framework that connects the IMU measurements with the forces applied on a sediment grain moving on a riverbed. IMUs were embedded a spherical and an ellipsoidal grain and used in flume experiments in which flow was increased until the grain moved. Acceleration measurements along three orthogonal directions were then processed
- 10 to calculate the threshold force for entrainment resulting in a statistical approximation of inertial impulse thresholds for both the lift and drag components of grain inertial dynamics. The ellipsoid IMU was also deployed in a series of experiments in a steep stream (Erlenbach, Switzerland). The inertial dynamics from both sets of experiments provide direct measurement of the resultant forces on sediment particles during transport which quantifies: a) the effect of grain shape; and b) the effect of varied intensity hydraulic forcing on the motion of coarse sediment grains during bed-load transport. Lift impulses exert a significant
- 15 control on the motion of the ellipsoid across hydraulic regimes, despite the occurrence of higher magnitude and duration drag impulses. The first order statistical generalisation of the results suggests that the kinetics of the ellipsoid are characterised by low or no mobility states and that the majority of mobility states are controlled by lift impulses.

1 Introduction

Q

River sediment transport is a critical process in landscape evolution (Tucker and Hancock, 2010), controls river morphology
 and ecology (Recking et al., 2015) and affects river engineering (Van Rijn, 1984). The study of the two-way relationship between transport processes and the corresponding morphology has a long history (e.g. Gilbert and Murphy, 1914), using approaches and mathematical conceptualisations that range from deterministic (e.g. Gilbert and Murphy, 1914; Shields, 1936; Ali and Dey, 2016) to probabilistic (e.g. Einstein, 1937; Grass, 1970; Ancey et al., 2008).

- Fluvial sediment transport is a complex two-phase flow defined by: (a) hydraulics (Kline et al., 1967; Nelson et al., 1995;
 Papanicolaou et al., 2002); (b) sediment properties and arrangement (Ashida and Michiue, 1971; Komar and Li, 1988; Kirchner et al., 1990; Buffington et al., 1992; Hodge et al., 2013; Prancevic and Lamb, 2015); (c) flow history across time scales (Shvidchenko and Pender, 2000; Diplas et al., 2008; Valyrakis et al., 2010; Phillips et al., 2018; Masteller et al., 2019; and, (d) biological and chemical processes that can rearrange or stabilise sediment (Johnson et al., 2011; Vignaga et al., 2013; Johnson, 2016).
- 30 To analyse the motion of a grain resting on a riverbed that is sheared by a turbulent flow (Dey and Ali, 2018), a large group of laboratory and theoretical studies use an implicit (fixed) reference frame. Historically, such analyses have been deterministic (implementing a single threshold shear stress or force at which grains are entrained (Gilbert and Murphy, 1914; Shields, 1936; Yalin, 1963; Iwagaki, 1956; Ikeda, 1982; Dey, 1999)). However, stochastic descriptions arguably capture better the complex
- particle-fluid interplay since near-bed turbulence which drives gain motion is inherently stochastic (Einstein, 1937; Grass, 1970;
 Papanicolaou et al., 2002; Marion and Tregnaghi, 2013). Coupled with advances in monitoring techniques (e.g. Papanicolaou
- et al., 2002; Fathel et al., 2016), stochastic treatments have led to Lagrangian, primarily numerical, formulations being applied to the full range of motion (McEwan et al., 2004; Bialik et al., 2015). The term Lagrangian means that sediment flow is observed from the perspective of individual mobile sediment grains and not a fixed time and space domain (as in traditional Eulerian approaches using fixed x,y,z co-ordinates, Ballio et al., 2018). The turbulence impulse approach (Diplas et al., 2008;
- Valyrakis et al., 2010; Celik et al., 2010) accounts for both the magnitude and the duration of the hydraulic forcing and is often categorised as a stochastic approach (Dey and Ali, 2018) since the stochastic nature of local turbulence is accounted for by the integration of turbulent forces acting on the grain over time. However, the grain forces are treated deterministically and through a detailed treatment of the force balance during incipient motion. Finally, the spatio-temporal approach (Coleman and Nikora, 2008) is different as the equations of motion are applied separately for the fluid (in a spatially averaged domain) and the sediment particles, linking the mode of transport with the scales of turbulence (Bialik et al., 2015).

Field experiments tracing individual sediment grains are in principle Lagrangian (Hassan and Roy, 2016). Lagrangian an-alytical models have been developed following advances in monitoring techniques that allow tracking of individual grains,
including magnetic (e.g. Schmidt and Ergenzinger, 1992; Hassan et al., 2009) and RFID tracers (e.g. Schneider et al., 2014; Tsakiris et al., 2015). An important milestone in the development of Lagrangian approaches for sediment transport was the
introduction of Discrete Particle Modelling techniques in simulations (McEwan et al., 2001, 2004; Schmeeckle and Nelson, 2003) which opened up the prospect for upscaling the Lagrangian metrics.

Lagrangian measurements find direct application in coarse grain gravel bed and bedrock river environments (e.g. Hassan et al., 1992, 2009; Ferguson et al., 2002; Hodge et al., 2011; Liedermann et al., 2012) and the morphological impact of Lagrangian dynamics in those environments is pronounced (Hodge et al., 2011). For example, the inertia of the typically larger particles transported in these streams has been identified as one of the factors contributing to non-accurate predictions of transport rates (Buffington and Montgomery, 1997; Bunte et al., 2004; Singh et al., 2009). Equally important is the lack of information on the energy transfer between these large particles and the river bed, particularly during impact. Recent experiments (Gimbert et al., 2019), show how this energy transfer can be inferred from seismic measurements, opening the way for

testing hypotheses that relate to river reach scale processes (eg. Burtin et al., 2014). Finally, for a complete understanding of these interactions to rotational component of grain movement cannot be ignored (Niño and García, 1998).

A particular advance in monitoring technology has been the development of sediment grain - scale inertial sensors which record at high frequency the accelerations and angular velocities experienced by grains during entrainment and motion (Kularatna et al., 2006; Akeila et al., 2010; Frank et al., 2015; Maniatis et al., 2013; Gronz et al., 2016; Maniatis et al., 2017). These applications became possible after the development of compact MEMS (Micro Electrical Mechanical Sensors) Iner-

65 tial Measurement Units (IMUs), assemblies of 3D MEMS accelerometers and 3D MEMS gyroscopes, which overcome many technical difficulties posed by older instrumentation (mainly caused by limited storage capacity, e.g. Ergenzinger and Jupner, 1992; Spazzapan et al., 2004). The focus of those works is the development of an IMU based sensor assembly (IMU enclosed in a grain or a purpose specific grain shaped artificial enclosure) that can successfully measure grain dynamics.

MEMS-IMU sensors measure the acceleration and angular velocity of the grain, which can be used to calculate the net force

70 acting on the grain. For the most accurate measurement of this force, sensors should be located at the grain's centre of mass. Data collected from within grains undergoing transport has potential to describe the timing of motion, forces acting on the grain and grain location. As a grain moves, its centre of mass moves and so the reference point for the force measurements is mobile. The latter means that the IMU measurements need to be transformed to a frame of reference that can be understood by an observer. Generally, an IMU accelerometer is a non-inertial frame fixed within the mobile body frame of the sensor

75 assembly.

60

In theory, the accelerations recorded by the IMU could be integrated to calculate grain velocity and integrated again to reveal location, a process referred to as dead-reckoning. One long-term goal for this approach would be to use IMUs to track a large number of grains through fluvial systems. However, real fixed ('strap-down') IMUs based on MEMS are not suitable for these integrations since the data contain several sources of uncertainty including signal noise and nano-scale mis-alignment of sensor

- axes. With sensors that are cheap enough to be deployed in large numbers, the accumulation of errors means that they cannot be used for 3D tracking of long term unconstrained motions (Woodman, 2007; Kok et al., 2017; VectorNav, 2016). This problem is well known in the fields of navigation and electrical engineering and the modelling of IMU errors is a significant research area (Zekavat and Buehrer, 2011) since the applications of this technology are numerous (Gebre-Egziabher et al., 1998; Grewal et al., 2007).
- 85 Despite the limitations, IMU sensors have been considered to be a suitable technique for measuring grain motion (e.g. Gronz et al., 2016). Recently Gimbert et al. (2019) used accelerometers to measure particle bed impacts in order to complement seismic measurements. However, for field deployments, relevant sensors have only been used as start and stop motion sensors (Olinde and Johnson, 2015).
- ę

The first goal of this paper is to introduce a simple rigid body model that connects measurements derived from an idealised IMU with existing models for grain motion. For this model to be successful, it is necessary to map the IMU body frame dynamics to the reference frame of motion (flume or riverbed). This resolution allows the inertial measurements to be related to the forcing on the particle and defines an explicit and unambiguous threshold for particle motion. The second goal is to introduce the calculation of inertial impulses above the drag and lift thresholds of motion, following the example of Diplas et al. (2008), for grain entrainments and short transport events. The calculations are performed for observations made in a set of flume entrainment experiments using two sensor assemblies: one spherical and one ellipsoidal. We apply the same analysis to

a time series of successive transport events measured with the ellipsoid sensor in a steep Alpine river (Erlenbach, Switzerland), calculating the force regime and the generated impulses during grain motion. Finally, we discuss how the combined dataset (flume and field experiments) for the ellipsoid sensor can be used for bootstrap calculations leading to the generalisation of the derived measurements.

100 2 Force measurements and a Newton - Euler regime for sediment motion

An IMU accelerometer records accelerations of the grain, which can be converted to forces acting on the grain by multiplying by grain mass. However, those accelerations are both fixed (e.g due to gravity) and variable (e.g due to fluid forces applied on the particle). The measurements are difficult to interpret because fixed and variable accelerations cannot be decoupled after the accelerometer begins to move and these accelerations are perceived differently from different reference frames. If the

sensor is static and gravity is recorded from the reference frame of the sensor it is then possible to derive a reliable orientation measurement. If the sensor moves and the sensor frame accelerations are compensated for gravity (by removing gravity from the raw accelerometer measurement) then the sensor will record the 3D components of the resultant or net force that mobilises?
the particle. This resultant is the force that can be observed from sensor who is static in relation to the particle. In Appendix A, we provide a technical explanation of the differences between non-inertial (mobile sensor frame) and inertial (observed from an external to the sensor fixed frame) accelerations and demonstrate the necessary transformations.

- The physics of IMU sensors define also the main difference between IMUs and other force sensors that have been used to monitor grain motion, such as load cells (e.g. Schmeeckle et al., 2007 and Lamb et al., 2017). Load cells measure the total vertical and horizontal forces applied to the grain prior to the onset of motion, but their application is also limited because they prevent the grain from being fully entrained. In the the case of a grain sitting on a horizontal because subjected
- to an increasing lift force, a load cell would record an increasing force up until the point at which the lift force was equal to the weight and the grain was entrained. In contrast, the gravity compensated IMU will record 0 until the point of entrainment after which the grain starts to accelerate. In flowing water, grains do not transition instantly from being completely stationary to being fully entrained, and the IMU will also record the movement of grain vibrations prior to entrainment. Consequently, load cells and IMUs measure different accelerate?
- 120 prioto entrainment, i prioto entrainment

The following derivations rely on the transformation of the IMU accelerations from the mobile reference frame of the monitored particle (frame b) to a local static frame where the applied hydraulic, particle and gravitational forces are analysed (frame r hereafter). The x and y directions of the r frame (r_x and r_y hereafter) are, respectively, parallel and perpendicular to

125 the mean direction of the flow and the river or flume bed. The z direction (r_z hereafter) is normal to the rection R = rection R
for the initial alignment of frames b and r (Appendix A). b - r transformations rely on the successive tracking of the relative orientation between the two frames which are represented here using quaternions (Appendices ABA = 0).



Figure 1. Forces in the Newton-Euler model. The diagram show the linear forces applied on the centre of the mass of the target particle. F_{Dnet} is the net drag force, F_{Lnet} is the net lift force and F_{cos} is the force of gravity (equation 1) all analysed in the local static frame r. The rotational component (torques generated by the surface force f_{rot} around the centre of the mass of the target particle) is analysed on the body frame of the particle (b, here depicted as aligned with the static frame r, the general case is discussed in Appendix A). ϕ is the downstream slope. For the results presented in this work r_x is the downstream direction. Also, for the flume experiments there is no cross-stream slope ($\theta = 0$)

Combined measurements of grain acceleration and angular velocity allow direct calculation of the forces and turning mo-130 ments acting at the grain's centre of mass. This type of model formulation is the Newton-Euler model in the rigid body

5

dynamics literature (O'Reilly, 2008). For a spherical particle resting on an inclined bed, irrespective of the degree of exposure to the flow, the Newton-Euler regime is defined as follows:

$$\mathbf{P}_{F_{net}} = \mathbf{f} + \mathbf{F}_{Gr} = m \begin{bmatrix} \mathbf{a}_{rx} \\ \mathbf{a}_{ry} \\ \mathbf{a}_{rz} \end{bmatrix}$$
(1)

$$\mathbf{Q}\mathbf{P}_{net} = f_{rot} \mathbf{P} = I_{cm} \begin{bmatrix} \mathbf{\alpha}_{bx} \\ \mathbf{Q}_{aby} \\ \mathbf{\alpha}_{bz} \end{bmatrix}$$
(2)

135 F_{net} and T_{net} are net forces and torques applied on and around the centre of mass of the particle. For the right hand elements of equations 1 and 2, m is the particle mass and I_{cm} is its moment of inertia (Appendix C). a_{rx} , a_{ry} , a_{rz} are linear accelerations resolved in the static frame r (Figure A1) measured using accelerometer measurements and equation A4. α_{bx} , α_{by} , α_{bz} are rotational accelerations resolved in the mobile but fixed on the particle frame b such that:

$$\begin{bmatrix} \alpha_{bx} \ \alpha_{by} \ \alpha_{bz} \end{bmatrix}^T = \begin{bmatrix} \frac{d\omega_{bx}}{dt} & \frac{d\omega_{by}}{dt} & \frac{d\omega_{bz}}{dt} \end{bmatrix}^T$$
(3)

140 where ω_{bx}, ω_{by} and ω_{bz} are angular velocities within the body frame reference frame as recorded by gyroscope measurements.

The left hand elements of equations 1 and 2, describe the forces applied on the particle. Hereafter, the lower case vectors and scalars (e.g terms f and f_{rot}) will refer to the interactions between hydraulic forces (turbulence) and particle forces (support forces and friction) that are not measured directly by an IMU and which cannot be decoupled using IMU measurements. The

145 vector f is the linear component of those interactions, applied on the centre of mass of the particle. The vector f_{rot} is the component of those interactions applied on the surface of the particle and it generates the torques around the centre of its mass (tangential to the particle's radius). The vector F_{Gr} defines the force of gravity rotated in the r frame and compensated for hydrodynamic effects (e.g buoyancy) as:

$$\boldsymbol{F_{Gr}} = R(_i^r q) \boldsymbol{W_{si}} f_v = \begin{bmatrix} \boldsymbol{F_{Grx}} \\ \boldsymbol{F_{Gry}} \\ \boldsymbol{F_{Grz}} \end{bmatrix}$$
(4)

The matrix $R(_i^r q)$ denotes the orientation of the slope in 3D (the relative orientations between the frames *i* and *r*, equation A5). W_{si} is the immersed weight of the spherical particle equal to $m_b g$, where m_b is its immersed mass (Papanicolaou et al., 2002) and *g* is the acceleration of gravity, both acting at its centre of mass. W_{si} has a constant direction in frame *i* and is rotated in *r* using $R(_i^r q)$. $f_v = 1 + [0.5\rho/(\rho_s - \rho)]$ accounts for the hydrodynamic mass effect (Papanicolaou et al., 2002; Celik et al., 2010) and ρ , ρ_s are the densities of the water and the particle, respectively. For our ellipsoid, we calculate F_{Gr}

assuming a sphere with the same volume as the ellipsoid, since resolving the hydrodynamic mass coefficient for an ellipsoid in 3D is beyond the scope of this work. The magnitude of F_{Gr} can be measured by deploying an IMU as an orientation sensor (Appendix A1).

The magnitude of F_{Gr} components along the drag direction (r_x) is given by the vector:

$$FG_D = F_{Grx}$$
(5)For the direction normal to the bed $(r_z, \text{ direction of lift hydraulic force) the } F_{Gr} \text{ component is the vector:}$ $FG_L = F_{Grz}$ (6)Similarly, we can use the linear accelerations of equation 1 to separate the components of the net force (F_{net}) along the drag and lift directions as: $F_{Dnet} = m a_{rx}$ (7)

165 and

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$$F_{Lnet} = m \, a_{rz} \tag{8}$$

2.1 Inertial measurements and the threshold of motion

The linear force threshold of motion is defined for $a_r = \begin{bmatrix} 0 & 0 \end{bmatrix}^T$, representing the explicit state where the forces are balanced (a resultant force F_{net} of equation 1 equal to 0) and the particle is completely immobile. Since all the linear forces are transformed in the *r* frame, exceeding this threshold relates to either a motion along the r_x drag direction (threshold defined at $F_{Dnet} = 0$) and/or a motion along the r_z lift direction (threshold defined at $F_{Lnet} = 0$). Given the force balance of equation 1, the threshold of motion is also the point where the resultant of the combined forces represented by f (which includes the

hydraulic lift and drag forces) balance the rotated gravity forces. For the drag direction, the component of f is given by the vector:

$$175 \quad f_D = F_{Dnet} - FG_D$$

180

(9)

and the threshold of motion is at the point where $f_D = -FG_D$

If F_{Dnet} exceeds 0 then f_D exceeds the component of gravity FG_D towards the positive (+) drag direction (downstream). Consequently, the condition $F_{Dnet} > 0$ is the exact equivalent of the condition $f_D > -FG_D$. We use the second description (for both the drag and the lift components) in order to highlight the fact that the f interaction primarily balances the components of gravity defined by the mean orientation of the bed.

For the lift direction, the component of *f* is given by:

$$f_L = F_{Lnet} - FG_L \tag{10}$$

Similarly to the drag components, the condition F_{Lnet} \Re is the exact equivalent of the condition $f_L > -FG_L$ (the particle moving towards the r_z positive direction, upwards).

- The rotational threshold of motion is defined by the state where the balance of torques around the centre of mass of the particle are balanced ($T_{net} = 0$). In the Newton-Euler model introduced here, the sum of torques captures a spinning rotational component defined by the product of the moment of inertia of the rotating particle (I_{cm}) and the body frame angular accelerations (equations 2 and 3). It is important to note that this rotation differs from the *orbital* rotation defined around the centre of mass of a supporting particle which is a common description in the hydraulic literature (e.g. Papanicolaou et al., 2002).
- 190 The torques are analysed in the body frame of the particle (b) and a non-directional description of the rotation threshold is given by the norm of angular accelerations ($\|\alpha_b\|$) exceeding 0. After equations 2 and 3 the critical condition for particle rotation is given by:

$$T_{net} = I_{cm} \|\boldsymbol{\alpha}_{\boldsymbol{b}}\| = I_{cm} \|\frac{d\boldsymbol{\omega}_{\boldsymbol{b}}}{\boldsymbol{\varphi}_{lt}}\| = f_{rot} R \ge 0$$
(11)

It is useful to re-state here that all the derivations so far are for a spherical particle (the equivalent of equation 11 for an 195 ellipsoid is given in Appendix C). Using equation 11, it is possible to estimate the magnitude of the tangential component f_{rot} (Appendix C). The calculation reveals that, for the scale of the particles discussed here, the effect of the tangential force, in terms of force magnitude applied on the particle, is negligible in comparison to the linear forces. This is explained by the dependency of the sum of torques (T_{net}) on the moment of inertia I_{cm} of the particle which is generally a very small number even for relatively coarse particles (e.g. for the spherical particle introduced in section 3 it is equal to 0.00085 kg.m²). For this 200 reason, we will only focus on the linear net force and the interaction f which we calculate as the difference between the net force and the gravitational components. However, we demonstrate the scale difference between the rotational and the linear components of particle motion in Figure 3 and Appendix C.

2.2 General kinematics and impulse

For completeness we note that if $a_r = [a_{rx} a_{ry} a_{rz}]^T$ is the 3D vector of linear acceleration of equation 1 in the r reference frame, grain linear velocities can be calculated by single integration. The velocity in the r frame is $v_r(t) = v_r(0) + \int_0^t a_r(t) dt$) and the total kinetic energy can by calculated from these velocities as $K = \frac{1}{2}m||v_r||^2 + \frac{1}{2}I_{cm}||\omega_b||^2$ with $\frac{1}{2}m||v_r||^2$ being the translational and $\frac{1}{2}I_{cm}||\omega_b||^2$ the rotational component. The condition K = 0 represents the threshold of rolling for any rigid body. However, this is not a direct equivalent to the typical rolling mode of entrainment because of the differences in the definition of rotation (spinning vs orbital) and the factor that there here is no assumption about the slipping condition as K can

210 be used to describe a rolling with slipping motion.

To account for both the duration and the magnitude of a force, the impulse for duration δt starting from the time t_i is defined as:

 $\int F(t)dt$

(12)

The subsequent analysis focuses on the calculation of impulses for specific time durations δt (Diplas et al., 2008; Celik

et al., 2010; Valyrakis et al., 2010). We calculate the impulse of the resultant interaction f above the threshold of motion and towards the positive drag and lift direction. This happens when the balancing gravity components F_{Gr} (equations 5 and 6) are exceeded, thus for the conditions :

$$f_D > -FG_D \tag{13}$$

(14)

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220 f_L>-FG_L
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which, as explained above, can only be satisfied when the magnitude of the net force exceeds $0 (F_{net} > 0)$, net force measured using gravity compensated accelerometer data, equation A4). In practice, we only account for the net forces that exceed the noise threshold of the accelerometer sensor (Figure 3, YEI, 2014; Maniatis, 2016). Finally, it is important to note that the calculated impulses are transferred to the particle from fluid turbulence and coherent flow structures, however this transfer is not described in this work. Here, the impulses capture directly the flow-particle interaction. The terms impulse, net impulse and inertial impulse are used interchangeably hereafter and they relate to impulses of the non zero net force.

3 Laboratory and field experiments

Two sensor assemblies were deployed, one sphere and one ellipsoid (described in Maniatis, 2016). The 90 mm diameter, 1.019 kg, sphere is solid aluminium with a symmetrical cavity for the IMU centred at the origin of the sphere. The ellipsoid (axes 100, 70, 30mm), made of the same material, weighs 0.942 kg. The cavity in the ellipsoid was designed to ensure that the 230 IMU axes align with the principal axes of the whole device. The density of both devices after the cavity cut is 2670 ± 3 kg.m⁻³ approximating the density of quartz (2650 kg,m⁻³). The measuring unit is the TSS-DL-HH-S sensor from Yei-TechnologiesTM (YEI, 2014), equipped with a gyroscope ($\pm 2000^{\circ}$, s⁻¹ sensitivity) and an accelerometer with a maximum range of ± 400 g. The acceleration range is one of the main reasons for the selection of this IMU as lower range accelerometers, particularly those in the very common ± 20 g range, are not suitable for capturing the forcing in natural environments (Maniatis et al., 2013). The 235 factory maximum sampling frequency is 250 Hz. The nominal sampling frequency of the sensor (used for all the flume and field experiments presented in this work) is 50 Hz which permits constant use for approximately 5 hours (LiPo rechargeable battery). This frequency was chosen after considering displacements of 15 particle diameters per second reported by Drake et al. (1988). This is adequate for capturing the dynamics of the particles if collisions and strong interactions with the bed are excluded (different type and higher frequency piezoelectric sensors are more suitable for the full measurement of impacts). 240 The measuring unit was calibrated through a series of shaking table and and rolling drop experiments which are described in Maniatis (2016, Chapter 6), along with the corresponding filtering workflow. During this calibration the noise threshold for the accelerometer was defined at 1.1 ± 0.23 N and the value of 1.2 N is used for the following presentation. The gyroscope has a

245 3.1 Laboratory entrainment experiments and separation of impulsive events

All flume experiments took place in an ArmfieldTM flume of 7 m effective length and 0.9 m internal width and at a fixed bed slope of S = 0.02. This slope corresponds to an orientation by the quaternion $_{i}^{r}q = [0.92, 0.14, -20, 028]$ (Appendix A). $_{i}^{r}q$ was measured by aligning the X-IMU axis with the centreline of the flume, the Y-IMU axis with the transverse direction and the Z-IMU axis with the direction normal to the bed. The positive X-IMU axis coincides with cross-section averaged flow direction.

- A bed of plastic hemispheres of the same diameter (90 mm) as the spherical device was constructed. The hemispheres were glued to form a 0.5 m (L) x 0.9 m (W) section and the whole section was placed at the point which allowed for the test particle to be at 4.5 m from the upstream boundary of the flume. A thin layer of 1.5 mm uniform sand was glued to both the hemispheres section and the upstream flume surface. The section upstream of the hemispheres was filled with very densely packed non-uniform, rounded gravel ($D_{50} = 0.015$ m) enabling the development of turbulent flow. No sediment transport occurred from
- the upstream gravel section during the entrainment experiments. The hemispheres were glued in positions that produced a 0.045 m protrusion of the sphere above the top of the hemispheres, although the sand layer made the protrusion higher, by partially filling the gaps between the hemispheres, and non uniform around the test particle (≈ 0.050 m). The ellipsoid was only supported by the hemispheres and was fully exposed to flow (Figure 2).
- For the experiments, the spherical device was placed on the flume centreline in a saddle position between four bed hemispheres and the three sensor axes were aligned with the inertial frame $i ({}_{i}^{b}q = [1,0,0,0]$, Appendix B). After positioning the sensor, the discharge was increased at a constant rate of 0.028 1.s⁻² until the particle was entrained. Acceleration and rotation were measured for the duration of the flow increase, throughout the sensor movement, and for 10 further seconds after it stopped moving. All the experiments were videoed at 60 fps using a standard GoPro Hero 7. The same experimental protocol was followed for the ellipsoid device, differing only in that the particle was initially aligned to the frame of reference of the flume bed (${}_{i}^{b}q = {}_{i}^{r}q = [0.92, 0.14, -20, 028]$). This resulted in the X - sensor long axis coinciding with the X flume direction (the direction of the flow, Figure 2).

Ten entrainment experiments were conducted with each device. We define the entrainment point as the initiation of the vibration that dislodges the particle by one particle diameter. For the ellipsoid, entrainment is defined as the initiation of the vibration that dislodges the particle by one b-axis. Drag and lift forces as well as the duration and the inertial impulses for cases 270 where the grain started to move ($f_D > -FG_D$ and/or $f_L > -FG_L$) were calculated using the derivations of section 2.1. For the spherical sensor, the gravitational components are $FG_D = 3.99$ N and $FG_L = -7.25$ N, respectively (Figure 3). The equivalent values for the ellipsoid, using the geometry of a sphere of equal volume, were $FG_D = 5.11$ N and $FG_L = -9.28$ N. The critical discharge for the sphere was $24.8 \pm 1.8 \, 1.8^{-1}$ which corresponds to a critical depth of 0.095 ± 0.015 m, measured from the top of the supporting hemispheres. The critical discharge for the ellipsoid was $45.2 \pm 2.21.s^{-1}$ which corresponds to a critical depth of 0.12 ± 0.02 m ($\tau_* = 0.01$ and 0.02 respectively, $\tau^* = \frac{HS}{(\rho_P - \rho_f)D}$, Appendix D). The hydraulic parameters at mean critical discharges are calculated in Table D1. The flume experiments have particle diameter to flow depth ratio close to

 $1 (d/H \approx 1)$ despite the fact that the particles were fully submerged at the critical depth for all the experiments (Figure 2). The

tested conditions are relevant to the entrainment of coarse particles in steep mountain streams but they should not be directly generalised to other bedload transport regimes despite the generality of the Newton-Euler model presented in section 2.



Figure 2. Laboratory setting and initial alignment The diagram shows the arrangement of the bed of hemispheres and the test position (4.5 m downstream the entrance of the flume). Upstream of the hemispheres bed, the flume was filled with densily packed gravel ($D_{50} = 0.015$ m, black line in the graph) to enable the development of the flow. The sphere diameter is 0.09 m, and ellipsoid b-axis is 0.07 m. The height of centroid above the crest of the glued hemispheres is approximately 0.02 m for the sphere and 0.03 m for the ellipsoid. The critical depths correspond to the hydraulics summarised in Table D1. *r* stands for river-bed (flume in this case) reference frame, *b* for body frame and *i* for the inertial reference frame ($_{i}^{r}q = [0.92, 0.14, -20, 028]$), Appendix A). *S* is the slope of the channel (0.02). The sketch also depicts the initial alignment for each device which (b - i for the sphere and b - r for the ellipsoid).

280 **3.2** Probabilistic impulse threshold for motion

Entrainment was observed independently from video recordings which were synchronised with the experiments from the start of the flow increase (section 3.1). These observations are used to calculate statisticallty the probability of entrainment as a function of the impulse of the interaction f exceeding graving rollowing the framework presented in Maniatis et al. (2017), the exact time of entrainment was noted in the video recording and the derived inertial impulses were separated into a binary, pre- and post-entrainment, data set. Logistic regression was used to describe the probability of entrainment, with Pr > 0.5defining the threshold of motion. Following the conceptualisation of Grass (1970), exceeding that threshold relates to impulses that are able to fully dislodge the particle, in contrast to the conditions that conceptualisations. The difference here is that this conceptualisation is applied to events in which impulses exceed the thresholds defined in equations 5 and 6). Video recording was not possible in the field setting, so this calculation is only presented for the laboratory experiments.

290 **3.3** Field testing

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Field experiments took place within a 5 m long straight and confined reach of the Erlenbach mountain stream in Switzerland, approximately 15 m upstream of the concrete channel section and 55 m upstream of the sediment retention basin in which continuous bedload transport measurements have been made during the past 30 years (Turowskiet al., 2011; Rickenmann et al., 2012). The stream has a step-pool morphology allowing the second be retrieved from pools, so the ellipsoid sensor was submerged on a bare bedrock section close to the edge of a step (average slope $\mathcal{P}= 0.1$, cross-averaged flow depth H =295 0.1 m, $\bar{\tau}_* = 0.095$), aligned to the same orientation as the riverbed $\binom{b}{i}q = \frac{r}{i}q = [0.50 - 0.39 \ 0.34 - 0.68]$, assumed parallel to the banks and the cross-averaged flow direction, at the approximate centre line of the stream) and allowed to be transported until it stopped moving and remained immobile for at least 10 seconds (Figure 3c). In all the experiments the sensor was entrained fully and immediately as there was no vibration *in situ*. The first one second of each transport event was removed from the data 300 as the effect of holding and releasing the sensor were still present. Ten transport events were recorded and processed similarly to the flume experiments. The corresponding $f_D > -FG_D$ and $f_L > -FG_L$ (Figure 3c) were calculated similarly to the flume experiments. The average travel distance for each transport event was 2 ± 0.43 m (from the point of release to the point of deposition, tape measurements) and the average event duration, after the first second of release, was 3 ± 0.6 seconds (Figure 3) \mathcal{O} constablish a representative orientation for the reach in relation to the orientation of gravity, the IMU was aligned parallel to the approximate centreline (X-IMU axis, X992 cross-section averaged flow direction), trasverse (Y-IMU axis) and normal 305 to the bed (Z-IMU axis) directions within the stream. The hydraulic parameters are summarised in Table Data



Figure 3. Example flume and field experiments (a, b) Calculated drag (a) and lift forces (b) for the same flume experiment using the spherical sensor. The vectors F_{Dnet} and f_D record the magnitudes of the net force and the resultant hydraulics- resistance interaction on the 2D-plane parallel to the flume bed. The vectors F_{Lnet} and f_L record the net force and the hydraulic resistance interaction, respectively, along the normal to the flume bed direction. The gravitational components ($FG_D = 3.7$ N and $FG_L = -7.25$ N) were calculated using equation to the flume bed line (t=10.1 s), shows the exact point of the full dislodgment of the particle as determined from the video recording. The *N*.*T* lines indicates the noise threshold of the accelerometer sensor, the calculated impulses concern sequences of point forces that exceed *N*.*T* (c) Net drag forces recorded during one field experiment (Erlenbach). The vertical line indicates the time of one second after the release of the sensor (see section 3.3). The calculation of the tangential force magnitude (f_{rots} and \mathcal{G}_{rots}) were and the ellipsoid, respectively is based on equations C1 and C5.

4 Results

The flume experiments demonstrate the differences between the spherical and the ellipsoid particle during incipient motion (Figures 4 and 5). For the sphere, drag and lift impulses over the gravity forces ($f_D > -FG_D$ and $f_L > -FG_L$) occur

- 310 for similar durations and generate impulses of similar magnitude ($f_D > -FG_D$ impulses median = 0.45 N.s, $f_L > -FG_L$ impulses median= 0.46 N.s). The relationship between the duration of exceedance events and the generated impulse follows an approximately linear trend, although variability is marginally higher for the relationship between drag impulses *I* and corresponding durations (*t*). For the relationship *I* vs *t*, $R^2 = 0.78$ (p-value < 6.52 x 10⁻¹⁶) for the drag events and 0.89 (p-value < 4.2 x 10⁻⁹) for the lift events, Figure 4a).
- 315 The results from the ellipsoid sensor demonstrate a strong influence of the lift forces. Exceedance impulses occur for similar durations and magnitudes, however there is a strong bias of the lift distribution towards the shorter and low impulse events. The drag duration and impulse distributions include more and higher magnitude outliers than the lift distributions (*f_D* > -*FG_D* impulses median = 0.08 N.s, *f_L* > -*FG_L* impulses median= 0.022 N.s). For the ellipsoid, the *I* vs t relationship has *R*² = 0.95 (p-value < 2.2 x 10⁻¹⁶) for the drag events and 0.67 (p-value < 2.2 x 10⁻¹⁶) for the lift events (Figure 4b). For all these 320 threshold exceeding events the sensor was vibrating until entrained as observed from both video and IMU data.
- Using the video recording observations, the impulse thresholds for entrainment were approximated with ogistic regression. The probability of entrainment as a function of impulse (Figure 5 (a) and (b)) highlights the control of short lift events on the entrainment of the ellipsoid. The impulse threshold for the sphere is close to 0, as all the approximated probabilities exceed 0.5. However, there is significant variability in this calculation (wide 95% confidence intervals) which indicates the wide range of impulses that can lead to entrainment of the sphere. In contrast, the entrainment of the ellipsoid demonstrates a dependency
- on lift impulses as the lift threshold is lower (ellipsoid lift impulse threshold = 0.27 ± 0.03 N.s) and the drag threshold is approximated with less confidence (ellipsoid drag impulse threshold = 0.74 ± 0.27 N.s).

Finally, the results from the field experiments (Figure 6) indicate similar statistical behaviour to the laboratory experiments but higher variability. Drag forces are of higher magnitude and duration than the lift forces ($f_D > -FG_D$ impulses median 330 = 0.13 N.s, $f_L > -FG_L$ impulses median= 0.08 N.s), but there is an abundance of low magnitude lift impulses that affect strongly the motion of the ellipsoid. In the Erlenback the duration of the exceedence events is also a proxy for the generated impulse with the *I* vs t relationship being more linear for the lift events compared to the drag events (*I* vs t $R^2 = 0.66$, p-value $< 2.2 \times 10^{-16}$, for the drag events and 0.88, p-value $< 2.2 \times 10^{-16}$, for the lift events).



Figure A Inertial impulses and duration of threshold exceedance events for laboratory experiments. For the spherical particle (a) the dra multiple median (median of $f_D > -FG_D$ events) is 0.01 N.s higher than the limit multiple median (median of $f_L > -FG_L$ events). For the ellipsoid (b) the equivalent difference is 0.013 N.s. The relationship between the duration (t) and the multiple (I) during the exceedence events is linear for both the sphere and the ellipsoid (and for both $f_D > -FG_D$ and $f_L > -FG_L$) events. The entrainment of the ellipsoid is more dependent on short and low lift implesses than the sphere, demonstrating the effect of shape on the inertial dynamics.



Figure 5. Probabilistic inertial impulse threshold for laboratory experiments. Logistic regression of the probability of entrainment for the spherical (a) and ellipsoid (b) particles. The calculation is based on the combination of video recordings and inertial impulse measurements during drag and lift threshold exceedance ($f_D > -FG_D$ and $f_L > -FG_L$ events). For the sphere there is little statistical difference between the calculated inertial impulses as over 95 % of the values relate to an entrainment event (the probability threshold 0.5 is always exceeded). For the ellipsoid, the probabilistic lift inertial impulse threshold relaxes to 0.27 N.s (blue vertical line, (b)) and the drag threshold relaxes to 0.74 N.s (red vertical line, (b)).



Figure 6. Inertial Impulses and duration of threshold exceedance events for field experiments. During short transport events (average travel distance = 2 ± 0.43 m) the drag inertial impulse median (median of $f_D > -FG_D$ events) is 0.05 N.s higher than the lift inertia impulse median (median of $f_L > -FG_L$ events). The relationship between the duration (t) and the inertial impulse (I) is linear for both for both $f_D > -FG_D$ and $f_L > -FG_L$ events. During *in-situ* transport the drag forces are of comparable magnitude and duration, however, short and low magnitude $f_L > -FG_L$ impulses have a strong influence on the motion of the ellipsoid.

5 Discussion

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335 5.1 IMU sensors and geomorphological applications

The advantage of using an IMU sensor for capturing grain motion is that the sensor solves a complex force and torque balance and removes any ambiguity in whether or not a test particle is in motion, as motion leads to the explicit thresholds F_{net} and/or T_{net} exceeding 0. Entrainment is captured directly and, assuming correct sensor calibration, robustly. IMUs can be a useful tool for geomorphologists since they offer a realistic prospect for monitoring particle motion during transport without invasive apparatus which is not possible with standard equipment, especially in field applications (e.g. PIT tracers). At the same time, it is important to recognise that that exceedance of the explicit thresholds above does not always produce complete dislodgment of the particle and also does not directly describe the modes of transport in the context that is commonly assumed for sediment hydraulics (e.g. differences in spinning and orbital rotations, section 2.1). For a complete understanding and effective prediction of grain motion both the hydraulic and the particle forces need to be measured, analysed and decoupled from the inertial forces

345 we measure in this study.

Further, there has been a recent rapid increase in use of IMU sensors, but most off-the-shelf IMU sensors are not suitable for the range of forces characterising natural sediment transport, especially if the focus is on particle interaction or impacts (Maniatis et al., 2013). In addition, the physics of IMU sensors are complex and a number of common assumptions about their use do not always hold. For example, while dead-reckoning appears to allow positions to be recovered by double-integration of

- 350 linear accelerations, uncertainties introduced during the production of IMUs (mostly nm scale imperfections on the alignment of the MEMS) lead to extreme uncertainty in positional estimates. A second issue involves calibrating IMU accelerometers which has often been done using free fall drop experiments. An accelerometer in free fall will measure 0 acceleration despite being subjected to the acceleration of gravity, as gravity in the context of the body frame of the accelerometer is a so-called fictitious force (Appendices A and B). Consequently, the force/impact results of a free fall drop experiment which relies solely
- 355 on IMU measurements are highly dependent on how quickly the sensor is programmed to enter and wake up from the free fall detection state (Clifford, 2006). It is possible to approximate the height of the free fall using the approximate time of the free fall state. However, the measurement of the impact force needs a very detailed description of both the impact surface and the low-level code that controls all the basic operations of the sensors (on-off routines, logging, storage-handling etc.). This low-level programming is a black box for proprietary off-the-shelf sensors and for users without suitable programming skills.
- 360 Finally, it is not possible to derive directional information, even for forces, for long mobile periods without complementary corrections (Kok et al., 2017) or without a detailed presentation of the reference frames involved and their initial alignment.

Here, we calibrated and deployed a commercial IMU sensor following standard procedures (Maniatis, 2016), but the precise corrections used are sensor specific and similar procedures should be followed again for any other IMU sensor. The calibration of force measurements is likely to be standardised and simplified in the near future as the use of IMU sensors develops further.

365 Similar standardisation for the direction of forces is potentially further away as it requires using IMU sensors that rely on optical technology and which are currently not manufactured with physical dimensions or within a price range that is accessible for sediment transport studies (De Agostino et al., 2010).

5.2 Relationship with previous work and first order statistical generalisation

Two aspects of this study are particularly important to address before we make comparisons with previous studies. The first is

- 370 that we made inertial measurements from within the sediment particles, which are fundamentally different from measurements of fluid turbulence that are often used for predicting sediment motion. The second is that the flow regimes under which we made measurements, with varying shallow flows, differ from those in many studies of sediment motion. Both of these aspects provide new insights into sediment movement, but they require care in making direct comparisons with studies that have used different approaches and/or hydraulic conditions.
- This work uses a theoretical framework which has the potential to enhance the mathematical modelling of sediment transport. The Newton-Euler model of section 2, in conjunction with the quaternion transformations of Appendix A, can be read as a 3D and unrestricted Lagrangian - Eulerian model for sediment transport. In our analysis, particle dynamics are transformed from a Lagrangian domain (and the mobile body frame of the particle b) to a static Eulerian domain (frame r) which is most commonly used for the analysis of turbulent flow. Ballio et al. (2018) analyse the topic in detail and provide a comprehensive
- 1-D Lagrangian Eulerian model which also accounts for the intermittency of sediment transport using a binary classification of mobile and non-mobile states. Our presentation can be used to define 3D Lagrangian dynamics, including rotation, in full and then to transform the corresponding kinematic properties to the Eulerian domain for direct comparison with the turbulent forces. We acknowledge that the verification of the 3D Lagrangian Eulerian model is heavily dependent on the inertial measurements, and particularly the constant tracking of relative orientation between the frames. However, it is possible to predict that future calibration experiments deploying IMUs will be used this way to parametrise simulations.
- Previous laboratory studies using fixed force meters attached to grains (Schmeeckle et al., 2007; Cameron et al., 2019) report nominal drag and lift forces of the order of 10^{-1} N for 0.008 - 0.025 m diameter articles, the measured force normalised by the submerged weight of the average diameter particle is of order 0.029 n the flume results presented here, the inertial drag and lift forces during entrainment are of order 10^{1} N, with normalised values for the sphere and ellipsoid being 15.6 and 17.1,
- 390 respectively. Differences in force magnitude are expected since: a) the inertial sensor is s freely mobile enabling the inertia of the moving particle to be recorded; and, b) the hydraulic regimes in both Schmeeckle et al. (2007) and Cameron et al. (2019) correspond to fully developed flows.

Static vibration sensors were also deployed by Lamb et al. (2017) who attached them to a wide range of test particles (D = between 0.075 and 0.218 m). They reported drag forces of order 10¹ - 10² N and and lift forces of order 10¹ N (drag forces

- 395 from 5.4 to 40.7 when normalised by the submerged weight of the average diameter particle). Also, the hydraulic regime used by Lamb et al. (2017) is comparable to both the laboratory and the field experiments presented here. The two studies yield very similar forces at entrainment, using different measurement methods, so validating the results from our inertial sensor. However, in general it is important to consider the type of sensor (static or restricted vs mobile), the data processing model and the experimental protocol (varied or steady flow) when different force measurements are compared.
- 400 Lamb et al. (2017) also observe a predominance of negative lift forces, especially for partially submerged particles, that may have significant morphological implications as potentially explaining, along with turbulence intensities, lower than predicted

sediment fluxes in steep mountain streams. In our work, the inertial negative lift forces are measured (Appendix E) but the exceedance events ($f_L > -FG_L$) are only calculated for the positive lift forces ($F_{Lnet} > 0$). The negative lift forces are components of the resultant force which can have a strong hydraulic component, as argued in Lamb et al. (2017), but they can also be a reaction to positive lift forces during the motion of the particle (and especially the motion of the ellipsoid, Figures 3

and E) which requires further investigation.

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The laboratory inertial impulse calculations demonstrate that, for unrestricted entrainments, there are observable differences between spherical and ellipsoid particles with the latter being more sensitive to the lift forces at entrainment threshold conditions. Those differences support previous work on the effect of shape on the response of particles in various hydraulic regimes

410 (e.g. Komar and Li, 1986; Demir, 2000) and on the mode of near-bed transport. Measurements of the effect of particle shape can now be made directly using inertial sensors.

The corresponding inertia calculations from the field also demonstrate that the ellipsoid is highly sensitive to lift forces and impulses. We observe higher mean magnitude lift forces on the ellipsoid in the field (0.62 N) than in the laboratory (2.13 N) (Figure E1). The greater negative lift forces during motion sugges that the particle has a reaction to the positive lift forces,

- 415 specifically those that exceed the threshold of motion and lead to transport. A similar increase in magnitude was observed for the drag components, with the instantenuous forces being up to 10 times the mean (Figure E1). The magnitude and duration of the exceedance events in the field are comparable to the observations in the laboratory experiments (Figures 4 and 6, the relationship between impulse and duration of events is more variable for the field experiments). However, the extreme forces are short lived and and so generate very small impulses close to 0 N.s. Differences in force magnitude and duration can
- 420 relate to transitions, as described by Shih and Diplas (2018), from hydraulic "impulse controlled" transport, as in our flume experiments, to "force-magnitude controlled" transport corresponding to the dynamics recorded in Erlenbach. However, an important difference between laboratory and field experiments lies in the scales of turbulence (Coleman and Nikora, 2008; Singh et al 2009), which requires further investigation since detailed flow measurements were not made during the presented experiments (eg. PIV measurements).
- 425 Overall, differences of particle inertial dynamics during grain entrainment and translation are important because they can potentially enhance predictions for grain particle travel distances with measurements from the field and particularly for large distances (Hassan et al., 1991, 2013). Measuring those differences is the most direct insight we can have for studying the effect of several morphological controls (eg. grain arrangement, burial depths, sediment sorting) until the high-frequency 3D measurement of tracer positions during transport becomes possible.
- comparison of the entrainment of spherical particles and relating them to critical thresholds (Diplas et al., 2008; Valyrakis et al., 2010; Celik et al., 2010; Valyrakis et al., 2010, 2011). In addition, there are recent efforts towards upscaling the effect of hydraulic impulses to fully developed bed-load equations (Shih and Diplas, 2018) and results pointing towards evaluating the morphological impact of different hydraulic impulse regimes (Phillips et al., 2018), highlighting the importance of deriving general statistical descriptions for grain inertial impulses.

Our data provide new insights into the roles of drag and lift impulses in entrainment. To begin to generalise these findings and to assess the interactions between lift and drag forces, a bootstrapping method is used here. We approximate the distributions of inertial lift and drag impulses for an ellipsoid particle and from a combination of laboratory and field measurements. This analysis is also the first step towards calculating the combined behaviour of the drag and lift distributions which can lead to the definition of joint distributions that have stronger explanatory and perhaps predictive value.

To combine the results from the laboratory and field ellipsoid experiments the impulsive exceedance events were normalised since the conditions are different for the laboratory and the natural conditions. Normalisation used the mean impulses for all drag and all lift forces, respectively. Also, the mean impulses were calculated separately for the laboratory and field experiments (Î = I/Īexp). After the normalisation, the laboratory and field results are combined into on clataset of normalised drag (Î_{Drag})
 and normalised lift (Î_{Lift}) impulses, which are assumed to be uncorrelated (Figure EQQQ)

The fitting of the representative distributions for \hat{T}_{Drag} and \hat{I}_{Qift} , permits bootstrap sampling from those distributions. Figure 7 shows 50000 random \hat{I}_{Drag} and \hat{I}_{Lift} combinations, sampled from the selected gamma and log-normal distributions $\hat{V}(f \otimes \hat{V}_{ag} \otimes f \otimes \hat{V}_{ag})$ and \hat{I}_{Lift} , respectively, Appendix F). After taking into account the normalised drag and lift impulse thresholds as

- defined in the Results (Figure 5), we conclude that the probability for the exceedance of the lift threshold is approximately 0.02, the probability for the exceedance of the drag threshold is approximately 0.005 and the probability of both thresholds being exceeded simultaneously is approximately 0.0001. The calculation confirms the observation that the transport of the ellipsoid particle is defined by states of zero or very low mobility (97.8% for the calculated combinations corresponds to dynamics that are below the normalised probabilistic impulse threshold for entrainment, Figure 5). 80% of the 1371 threshold exceedances of
- both thresholds. The calculation suggests that the majority of the mobility states of the ellipsoid will relate to the action of lift forces. The very small probability for the simultaneous exceedance of both thresholds is another possible effect of the particle's shape as spherical particles will protrude more and are more likely to be equally affected by both drag and lift components (Figure 5a). This type of calculations requires sample sizes that only advanced instrumentation, such as that presented in this work, can deliver. Similar frameworks can be used for meta analysis of existing results and to inform the design of future

460 experiments.

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Figure 7. Bootstrap normalised impulse sampling (lift and drag). \hat{I}_{Drag} and \hat{I}_{Lift} are fitted with a gamma and a log-normal distribution, respectively (Appendix F2). The normalised drag and lift thresholds (red dashed lines), are calculated using the probabilistic drag and lift inertial impulse thresholds presented in Figure 5 which were divided by the recorded mean inertial drag and lift impulse from the laboratory experiments.

6 Conclusions

This work introduces a framework that can be used to derive and interpret IMU measurements in sediment transport studies. The derivation of inertial measurements from mobile sediment grains requires a physical model that links the inertial dynamics with existing force (or moments) balance equations for sediment transport. The types of sensors, and associated smart pebble

- 465 assemblies currently deployed for the measurements of grain inertial dynamics are not suitable for 2D or 3D tracking of grain position. However, it is possible to measure net forces and impulses if the necessary transformations are applied consistent position. However, it is possible to measure net forces and impulses if the necessary transformations are applied consistent possible and laboratory measurements of inertial lift and drag impulses highlight the different entrainment behaviours of a spherical and an ellipsoidal particle. The lift net force is dominant during the unrestricted entrainment of the ellipsoid while there is no statistical difference between the effects of lift and drag mpulses on the entrainment of the sphere. Topped ag
- 470 component can be stronger during transport, however short impulses influen the motion of the ellipsoid significant

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640 Zekavat, R. and Buehrer, R. M.: Handbook of Position Location: Theory, Practice and Advances, vol. 27, John Wiley & Sons, 2011. Zhao, F. and van Wachem, B.: A novel Quaternion integration approach for describing the behaviour of non-spherical particles, Acta Mechanica, 224, 3091–3109, 2013. 645 *Code availability.* All the calculations were performed using the R-statistical software and specifically the libraries orientlib (Murdoch, 206) for the rotation calculations and fitdistrplus (Delignette-Muller et al., 2015) for fitting statistical distributions. Both libraries are open and freely available.

Author contributions. Georgios Maniatis performed the design and calibration of the sensor, the design and implementation of the experiments, all the physical and statistical calculations and produced the first draft of this paper. Trevor Hoey supervised the laboratory exper iments, reviewed several versions of the manuscript and contributed significantly to the interpretation and contextualisation of the results. Rebecca Hodge contributed to the design of the laboratory experiments, reviewed several versions of the manuscript and contributed to the design, supervised and assisted with the field experiments, reviewed several versions of the manuscript and contributed to the interpretation and contextualisation of the results. Alexandre Badoux contributed to the design of the field experiments and reviewed several versions of this manuscript.

655 *Competing interests.* The authors declare no competing interests

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Appendix A: Frames of reference, rotations and IMU measurements

To discuss the measurements recorded by an IMU, and particularly the measurements from an accelerometer and a gyroscope, it is necessary to introduce three basic frames of reference and select one of the many representations for arbitrary rotations in 3D.



Figure A1. Frames of reference *r* stands for river-bed (flume in this case) reference frame, *b* for body frame and *i* for the inertial reference frame.

The following assumptions and simplifications are used throughout this study:

- due to the small scale $(10^{-1} \text{ to } 10^{1} \text{m}, \text{typically})$ motion of sediment grains, an Earth frame (one that coincides with the inertial frame as defined below, but rotates with the Earth) is not defined. Also, the angular velocity of the earth (approximately $7.29 \cdot 10^{-5} \text{ rad.s}^{-1}$) is ignored.

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- for the same reason, the non-gravitational fictitious forces (such as the Coriolis effect) are ignored.

- = for the mathematical derivations, ideal IMUs (no error accumulation is considered) and perfectly aligned sensor assemblies are assumed. The errors associated with IMUs and especially with the magnitude of the integration errors are presented in relevant electrical engineering sources (eg. Kok et al., 2017).
- We define the body frame b as the coordinate frame of the moving IMU. For an ideal IMU the origin of this frame is located exactly at the center of both the accelerometer and the gyroscope and this center falls precisely on the center of mass of the complete sensor assembly (Maniatis et al., 2013, 2017). Frame b is mobile but fixed in the sensor assembly (fixed axes representation).
- The local geographical frame r is the stationary frame within which hydrodynamics are analysed. This is the reference frame used implicitly for all the single grain motion studies (Dey and Ali, 2018). For laboratory experiments, the r_x - r_y plane is exactly parallel to the flume bed and the r_z direction is normal to the bed. For the field experiments this alignment will be an approximation due to variations of the local topography. The inertial frame i is a stationary frame. Strap-down IMUs measure acceleration and angular velocity changes in response to this frame and its origin lies at the center of the Earth.
- Transforming information between these three reference frames is non-trivial. A widely-used method to represent the change between frame is the application of quaternions (Hamilton, 1844; Diebel, 2006). Quaternions are an extension of complex numbers used in the description of 3D mechanics, particularly 3D rotations. They are considered the most efficient description of unrestricted 3D rotations, as they are free from numerical errors that occur when other representations are used (such as the Gimbal Lock error associated with rotations expressed by Euler Angles, Appendix G2). A typical introduction to quaternions can be found in Valenti et al. (2015) and we follow that primer for a brief introduction to quaternion algebra in Appendix G1.
- 690 A unit quaternion ${}^{B}_{A}q$ defines a rotation from frame A to frame B and successive rotations are represented by quaternion multiplication. For each ${}^{B}_{A}q$, a Direction Cosine Matrix (DCM) $R({}^{B}_{A}q)$ is defined as a function of ${}^{B}_{A}q$ components (equation G11) which also represents a rotation from frame A to B. If ${}^{B}v$, ${}^{A}v$ are observations of the vector v in frames B and A respectively, they are related through the following typical matrix operation:

$${}^{B}v = R({}^{B}_{A}q)^{A}v \tag{A1}$$

If the frames A and B are relatively static (such as the inertial frame i in relation to the local geographic frame r) then both ${}^{B}_{A}q$ and $R({}^{B}_{A}q)$ are explicit. If B is rotating in relation to A (such as the body frame b in relation to the inertial frame i), ${}^{B}_{A}q$ and the corresponding $R({}^{B}_{A}q)$ need to be recursively updated. The transition quaternion \tilde{q} between two successive poses is defined by the applied angular velocity as:

$$\tilde{q} = \left[\cos\frac{\|\omega\|\delta t}{2}\sin\frac{\|\omega\|\delta t}{2}\frac{\omega_{bx}}{\|\omega\|}\sin\frac{\|\omega\|\delta t}{2}\frac{\omega_{by}}{\|\omega\|}\sin\frac{\|\omega\|\delta t}{2}\frac{\omega_{by}}{\|\omega\|}\right]^T \tag{A2}$$

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where $\omega_{bx}, \omega_{by}, \omega_{bz}$ are angular velocities observed along the b_x, b_y, b_z body frame axes respectively by the 3D gyroscope, $\|\omega\| = \|\omega_b\| = \sqrt{\omega_{bx}^2 + \omega_{by}^2 + \omega_{bz}^2}$ is the norm of angular velocities and δt the time of rotation, set here equal to the frequency of the IMU measurements.

equation A2 is part of the direct multiplication method (Whitmore, 2000; Zhao and van Wachem, 2013) and the updated quaternion ${}^{B}_{A}q'$ is derived as:

$${}^{B}_{A}q' = {}^{B}_{A}q \bigotimes \tilde{q}$$
(A3)

710 with the operation \bigotimes denoting quaternion multiplication (equation G4). After each update ${}^{B}_{A}q'$ is set as ${}^{B}_{A}q$. Inertial accelerometers measure the proper acceleration a_{b} applied within the body frame *b*. These accelerations will include gravitational acceleration, a uniform force in the inertial frame *i*. To derive the linear acceleration in the inertial frame *i* it is necessary to rotate the body frame measurement to the inertial frame and then to subtract gravitational acceleration. For this rotation the recursively updated $R({}^{i}_{b}q')$ DCM is used after calculating the ${}^{i}_{b}q'$ through equation A3. The linear acceleration in 715 the local geographical frame *r* is then given by:

$$a_r = R\binom{r}{i}q(R\binom{i}{b}q')a_b - g) \tag{A4}$$

where $a_b = [a_{bx} a_{by} a_{bz}]^T$ is the vector of the *b* frame accelerometer measurements, and $R(_i^r q)$ the explicit DCM that rotates the accelerations from *i* to *r* derived from the quaternion $_i^r q$. a_r defines the magnitude and direction of the resultant force or net force.

720 A1 Accelerometers as orientation sensors

In the equation A4, the raw accelerometer measurement is "compensated" for the action of the gravitational field in order to extract a measurement for the resultant force in the r frame. To explain the need for this calculation, it is important to note that accelerometers measure proper acceleration. Proper acceleration is different from coordinate acceleration which is generally defined as the rate of velocity (in a fixed inertial frame). In practice, if an accelerometer is placed on a flat surface will

- measure an upwards (positive) acceleration equal to 1g (9.81 m.s⁻²). In free fall, an accelerometer will measure 0 acceleration because in the non-inertial frame of the sensor there is neither a static (eg. gravity) nor a dynamic (motion or vibrational) force applied and the accelerometer will only "feel" the terminal velocity of the impact if it lands on a surface. When the accelerometer is subjected to an external force and it begins to move (in relation to a static inertial frame), there is no way to separate the components of the static and the dynamic forces, unless the relative orientation of the frames is constantly
- monitored (in the manner we describe in equation A4). At the same time, when the sensor is static, the non-compensated signal (the raw acceleration from the sensor) can provide an estimate for the relative orientation of the sensor's frame (b i or b r)

quaternions in relation to the frame that static forces are applied (gravity frame, *i*). Valenti et al. (2015), provide the solution for the calculating the ${}_{i}^{b}q$ quaternion from raw accelerometer measurements which, adapted to the notation used here is:

$${}_{i}^{b}q = \begin{cases} \begin{bmatrix} \sqrt{\frac{a_{bz}+1}{2}} & -\frac{a_{by}}{\sqrt{2(a_{bz}+1)}} & \frac{a_{bx}}{\sqrt{2(a_{bz}+1)}} & 0 \\ -\frac{a_{by}}{\sqrt{2(1-a_{bz})}} & \sqrt{\frac{1-a_{bz}}{2}} & 0 & \frac{a_{bx}}{\sqrt{2(1-a_{bz})}} \end{bmatrix}^{T} & , a_{bz} \ge 0 \end{cases}$$
(A5)

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In this work we used equation A5 to estimate the initial alignment of the frames for both the flume and the field experiments. In addition, the YEI sensor implements an onboard calibration routine which can verify the initial alignment using sensor fusion and a series of nonlinear filters (YEI, 2014).



Figure B1. Example incipient motion IMU data. Measurements from the incipient motion experiments using the spherical sensor. (a) unfiltered and uncompensated inertial acceleration measurements. The sensor is initially aligned to gravity which results in the *z* axis of the accelerometer measuring a mean value of 9.81 m.sec⁻². (b) angular velocity (rad.sec⁻¹) measurements derived from the gyroscope. (c) linear acceleration along the three body frame axis. This is the result of removing gravity from the inertial measurements shown in A and applying a FFT-high pass filter as described in Maniatis, 2016 (Chapter 6). (d) shows the kinetic energy calculation after integrating once the signal presented in (c) and applying the formula $K = \frac{1}{2}m||v_r||^2 + \frac{1}{2}I_{cm}||\omega_b||^2$ as described in section 29

740 The rotational component captured by the gyroscope relates to the moments of the forces applied on the surface of the measured particle via equation 2. For a sphere, f_{rot} is defined by rearranging equation 11 as:

$$f_{rots} = \frac{I_{cm} \|\frac{d\omega_b}{dt}\|}{R}$$
(C1)

where R is the radius of the particle. The relationship holds because the moment of inertia (I_{cm}) of the sphere is uniform. This implies that the shape of particle does not affect the direction of the resultant force. For the spherical sensor presented in this work the moment of inertia is calculated using the formula for a sphere $(I_{cm} = \frac{5}{2}mR^2)$ and is equal to 0.00085 kg.m².

For the ellipsoid, the same calculation is significantly more complicated. Firstly, the moment of inertia is not uniform. In this work, we implemented a numerical calculation during the design phase of the enclosure (using Solidworks, Système, 2016) for the principle axes of the ellipsoid. The principle axes coincide with the b_x , b_y and b_z body frame axes. Those are the equivalent of the axes a, b and c of the ellipsoid (where a = 0.1 m, b = 0.07 m and c = 0.03 m), they are fixed in the body frame and aligned

750 with the IMU axes (see Appendix A). The principle components of inertia for the ellipsoid sensor were calculated as: $I_{xx} = 0.00057 \text{ kg.m}^2$, $I_{yy} = 0.00060 \text{ kg.m}^2$ and $I_{zz} = 0.00094 \text{ kg.m}^2$. The non-principle components of inertia were calculated at the order of 10^{-8} kg.m² and they were ignored. The balance of moments for the principle axes is given by the following system of equations (often discussed as Euler equations, O'Reilly, 2008):

$$M_x = I_{xx} \frac{d\omega_{bx}}{dt} - (I_{yy} - I_{zz})\omega_{by}\omega_{bz}$$
(C2)

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$$M_y = I_{yy} \frac{d\omega_{by}}{dt} - (I_{zz} - I_{xx})\omega_{bz}\omega_{bx}$$
(C3)

$$M_z = I_{zz} \frac{d\omega_{bz}}{dt} - (I_{xx} - I_{yy})\omega_{bx}\omega_{bz}$$
(C4)

For the of the tangential force, a good approximation (ignoring the secondary moments of inertia) is given by dividing the principle moments by the half length of the corresponding principle axis of the ellipsoid giving:

$$f_{rote} = \sqrt{\left(\frac{2M_x}{a}\right)^2 + \left(\frac{2M_y}{b}\right)^2 + \left(\frac{2M_z}{c}\right)^2} \tag{C5}$$

760 The equations C1 and C5 were used to calculate the magnitude of the rotational component for the example experiments presented in Figure 3. Figure C1, shows the same calculations at a scale that reveals their fluctuation.



Figure C1. Force magnitude of the rotational component (a) Flume experiment (incipient motion) using the spherical sensor (Figure 3a). (b) Field experiment (Erlenbach) using the ellipsoid sensor (Figure 3e). The differences between the linear and the rotational componenents are between one and two orders of magnitude. f_{rot} is of the order between 10^{-3} and 10^{-2} N (sphere in the flume and ellipsoid in Erlenbach respectively) while the linear forces (applied directly on the centre of the mass of the particle) are of the order between 10^{0} and 10^{1} N (Appendix E)

Appendix D: Hydraulic parameters

Table D1. The parameters are estimated as follows: ρ_p/ρ_f is the ratio of an experimental particle density to fluid density ($\rho_f = 1000 \text{ kg.m}^{-3}$); Q is the flow rate; H is the flow depth (measured from the bottom of the bed to the water surface for Erlenbach and from the top of the hemisphere bed for the flume experiments); $U_b = Q/A$ is the bulk mean velocity (A is the cross sectional area of the flow); W is the channel width and S_b is bed slope. 0.105 (or 0.1) is also the average bedslope of the lowermost natural reach in Erlenbach of about 30 m length upstream of the stream again station; $F = U_b/(gd)^{0.5}$ is the Froude number; $R_b = (U_bd)/\nu$ is the bulk Reynolds number (where ν is the fluid kinematic viscosity at 0°Con Erlenbach and at 25 °C for the flume experiments). For the calculations referring to the ellipsoid, we assume D equal to the particle's h-axis

Experiment	ρ_p/ρ_f	Particle axis a (m)	Particle axis $b(m)$	Particle axis c (m)	Protrusion (m)	Q (1/s)	(m) <i>H</i>	W (m)	S_b	$U_b(m/s)$	Н	R_b	τ_*
Flume (sphere)	2.67	0.09	0.09	0.09	0.05	24.8	0.09	0.9	0.02	0.30	0.32	27555	0.01
Flume (ellipsoid)	2.67	0.10	0.07	0.03	0.03	45	0.12	0.9	0.02	0.41	0.38	50000	0.02
Erlenbach (ellipsoid)	2.67	0.10	0.07	0.03	0.03	120	0.15	3.5	0.1	0.22	0.19	34285	0.095



Figure E1. Histogram of inertial forces from all experiments. The inertial dynamics show that net lift (F_{Lnet}) and drag (F_{Dnet}) forces consistently fluctuate around zero. The vertical lines indicate the corresponding medians. The mean force magnitude for the sphere (in the lab) is $\bar{F}_{Dnet} = 0.57$ N and $\bar{F}_{Lnet} = 0.62$ N for the drag and lift directions, respectively. For the ellipsoid in the laboratory experiment the mean drag force magnitude is $\bar{F}_{Dnet} = 0.74$ N and the mean lift force magnitude is $\bar{F}_{Lnet} = 2.13$ N. Finally for the ellipsoid in Erlenbach mean $\bar{F}_{Dnet} = 1.22$ N and $\bar{F}_{Lnet} = 2.57$ N



Figure E2. Lift vs Drag force magnitude correlation (flume experiments). Regression analysis applied on the magnitude of calculated forces (drag and lift) a moderate correlation for the spherical particle (statistically significant Pearson's R = 0.3) and a weak correlation for the ellipsoid both in the laboratory and the field experiments (statistically significant Pearson's R = 0.046 for the lab measurements, Ellipsoid (Lab), and not significant Pearson's R = -0.00055 for the field ones, Ellipsoid (Erlenbach)). The latter supports the assumption of statistical independence between the two components for the ellipsoid, justifying the randomisation presented in section 5 (Figure 7). Pearsof R is an unbiased metric for this sample size.
Appendix F: Normalised impulse: selection of representative Drag and Lift distributions

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Three types of right tail distributions were considered (Cullen et al., 1999, F2) as good fitting candidates for both \hat{I}_{Drag} and \hat{I}_{Lift} : the Weibull, the gamma and the lognormal distributions. Goodness- of- fit analysis (Table F1), shows that \hat{I}_{Drag} is approximated marginally better by a gamma distribution (median shape = 0.529, median scale = 0.5) and the \hat{I}_{Lift} is approximated better by a lognormal distribution (median meanlog = - 0.66, median sdlog = 1.13).

Figure F1(a) shows the Cullen and Frey diagram for the identification of candidate distributions for the normalised Drag

- 770 Impulses (\hat{I}_{Drag}). Being in the "beta" region, the diagram indicates towards a right tail distribution (beta distributions are restrected between 0 and 1 and there is no physical relationship with the possible values for impulses). The skewness vs kurtosis relationship (blue dots), indicates a right tail distribution as a candidate as well. Figure F1 (b)-(e) show the graphical comparison between three candidate distributions (Weibull, gamma, and log-normal). Weibull and gamma distributions outperform the lognormal on the tails of the histogram (Q-Q plot), The median values are also captured better by the Weibull and
- 775 gamma distributions (P-P plot). Finally, the histogram and CDF diagrams confirm that the log-normal distribution is the least representative of \hat{I}_{Drag} . The gamma distribution marginally outperforms the Weibull distribution for \hat{I}_{Drag} both in graphical and goodness of fit comparisons and it is selected for the bootstrap calculation of Figure 7.

Figure F2 (a) shows the Cullen and Frey diagram for the identification of candidate distributions for the normalised lift impulses (\hat{I}_{Lift}). The skewness vs kurtosis relationship (blue dots), indicates a right tail distribution as a candidate. Figure F2 (b)-(e) show the graphical comparison between three candidate distributions (Weibull, gamma, and log-normal). The lognormal distribution outperforms the other candidates at the tail (Q-Q plot) and the median regions (P-P plot). Finally, the histogram and CDF diagrams confirm that the log-normal distribution is the best representative of \hat{I}_{Lift} and it is selected for the bootstrap calculation of Figure 7.



Oppose F1. Choice of distribution for drag impulses



Figure F2. Choice of distribution for lift impulses

Table F1. Fitted Distribution statistics

Drag Impulses - Statistics for fitted distributions

Goodness-of-fit statistics

	Weibull	gamma	Lnorm
Kolmogorov-Smirnov statistic	0.090	0.088	0.1084
Cramer-von Mises statistic	1.180	0.983	2.786
Anderson-Darling statistic	8.95	8.006	17.354
Goodness-of-fit criteria			
Akaike's Information Criterion	1311	1314	1359
Bayesian Information Criterion	1321	1323	1369

Lift Impulses - Statistics for fitted distributions

Goodness-of-fit statistics			
	Weibull	gamma	Lnorm
Kolmogorov-Smirnov statistic	0.069	0.090	0.019
Cramer-von Mises statistic	3.571	6.057	0.093
Anderson-Darling statistic	26.4	34.9	0.9
Goodness-of-fit criteria			
Akaike's Information Criterion	3968	4042	3576
Bayesian Information Criterion	3979	4054	3588

Table F2. Statistics for selected distributions

Drag Impulses - Statistics for selected distribution (Gamma)

Parametric bootstrap medians and 95% percentile CI

	Median	2.5%	97.5%			
shape	0.529	0.489	0.577			
rate	0.5	0.463	0.60			
Lift Impulses - Statistics for selected distribution (Lognormal)						
Parametric bootstrap medians and 95% percentile CI						
	Median	2.5%	97.5%			
meaning	-0.663	-0.710	-0.614			
sdlog	1.132	1.096	1.165			



Figure F3. Bootstrap parameters for selected distributions The graphs quantify the stability of the selected distributions for drag and lift impulses. For the gamma distribution (drag impulses) 1000 bootstrapped parameters were cross-compared, revealing a range of 0.1 for the shape parameter and 0.2 for the rate parameter (a). For the Logornal distribution (lift impulses) 1000 bootstrapped parameters were cross-compared, revealing a range of 0.15 for the log-mean parameter and 0.14 for log standard deviation parameter (b). The differences are marginal, indicating good stability of the selected distributions for the scaling of the data.

Appendix G: Quaternions and Rotations

785 G1 Summary Quaternion Algebra

Quaternions can be written in the form:

$$q = q_1 + q_2 i + q_3 j + q_4 k \tag{G1}$$

790 where q_1, q_2, q_3, q_4 are the components of quaternion q (and i, k, j unit imaginary numbers). **?** The quaternion conjugate is given by:

$$\bar{q} = q_1 - q_2 i - q_3 j - q_4 k$$
 (G2)

795 The sum of two quaternions is then:

$$\mathbf{Q} \qquad q + w = (q_1 + w_1) + (q_2 + w_2)i + (q_3 + w_3)j + (q_4 + w_4)k \tag{G3}$$

and quaternion multiplication is defined as:

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$$q \bigotimes^{w} = (q_1w_1 - q_2w_2 - q_3w_3 - q_4w_4) + (q_1w_2 + q_2w_1 + q_3w_4 - q_4w_3)i$$

 $(q_1w_3 - q_2w_4 + q_3w_1 + q_4w_2)j + (q_1w_4 + q_2w_3 - q_3w_2 + q_4w_1)k.$ (G4)

The quaternion norm is therefore defined by:

805
$$n(q) = \sqrt{\bar{q}q} = \sqrt{q_{Q}^2 + q_1^2 + q_3^2 + q_4^2}$$
 (G5)

With little manipulation, the quaternions can be directly related to four-element vectors. Quaternions can be interpreted as a scalar plus a vector by writing:

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$$q = q_1 + q_2 i + q_3 j + q_4 k = (s, \hat{v})$$
 (G6)

where $s = q_1$ and $\hat{v} = q_2 i + q_3 j + q_4 k$. In this notation, quaternion multiplication has the form:

$$q_1 \bigotimes q_2 = (s_1, \hat{v}_1) \cdot (s_2, \hat{v}_2)$$
815
$$= (s_1 s_2 - \hat{v}_1 \cdot \hat{v}_2, s_1 \hat{v}_2 + s_2 \hat{v}_1 + \hat{v}_1 \cdot \hat{v}_2)$$
(G7)

Finally, the rotation about the unit vector \hat{n} by an angle θ can be computed using the quaternion:

$$q = (s, v) = (\cos(\frac{1}{2}\theta), \hat{n}\sin(\frac{1}{2}\theta))$$
 (G8)

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The components of this quaternion are called Euler parameters. After rotation, a point p = (0, p) is then given by:

$$p' = qpq^{-1} = qp\bar{q} \tag{G9}$$

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since n(q) = 1.

A concatenation of two rotations, first q_1 and then q_2 , can be computed using the identity:

$$q_2(q_1p\bar{q_1})\bar{q_2} = (q_2q_1)p(\bar{q_1}\bar{q_2}) = (q_2q_1)p\overline{q_2q_1}$$
(G10)

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Finally, the transformation that gives the equivalent DCM for a quaternion $q = q_1 + q_2i + q_3j + q_4k$, is given by:

$$R(q) = [q_1^2 + q_2^2 - q_3^2 - q_4^2, 2(q_2q_3 - q_4q_1), 2(q_2q_4 + q_3q_1)$$

$$2(q_2q_3 + q_4q_1), q_1^2 - q_2^2 + q_3^2 - q_4^2, 2(q_3q_4 - q_2q_1)$$

$$2(q_2q_4 - q_3q_1), 2(q_3q_4 + q_2q_1), q_1^2 - q_2^2 - q_3^2 + q_4^2]$$
(G11)

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G2 The Gimbal lock (adapted from Maniatis, 2016)

To demonstrate the advantage of quaternions we rotate randomly the static vector of gravity. In an orthogonal Cartesian frame where the origin of the z-axis is the centre of the Earth, gravity is measured as $[G_x, G_y, G_z] = [0, 0, 9.81]$ m.sec⁻². If we assume a rigid body rotating freely and randomly in this frame we can do the rotation calculations. Avoiding further mathematisation, 840 the series of the calculations is the following:

- Randomisation of the body frame angular velocities of the rigid body $\omega_x, \omega_y, \omega_z$ in a $[-2\pi, 2\pi]$ range. A frequency of 100 Hz is used.
- Calculation of successive quaternions using direct multiplication for random angular velocities.
- Q Calculate the Direction Cosine Matrix from Euler angles. 845
 - = Rotate the vector of gravity in the body frame of the rigid body using both of the Direction Cosine Matrix using common matrix vector multiplication.

The vector expressed in the body frame is shown in Figure G1. The results are different and Gimbal lock (an inconsistent axis change when the second rotation approaches $\pm \pi/2$) occurs after the 450^{th} iteration which corresponds to 8 sec in simulation 850 time.



Figure G1. Random rotation of the static vector of Gravity. $[g_x, g_y, g_z] = [0, 0, 9.81] \text{ m.s}^{-2}$ in the gravity frame of reference as expressed in the body frame of randomly rotating rigid body (dt = 0.01 sec). (a) demonstrates the rotation calculations with the usage of Euler angles. Gimbal lock occurs after the 400 iterations. (b) shows the same rotation series calculated via quaternions. No Gimbal lock occurs and the result is easy to interpret as it is based on the use of the measured body frame angular velocities.